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**Inoue**

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(54) **BRUSH SEAL DEVICE**

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(\*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

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(21) Appl. No.: **14/424,294**

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(86) PCT No.: **PCT/JP2014/052779**

§ 371 (c)(1),

(2) Date: **Feb. 26, 2015**

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(30) **Foreign Application Priority Data**

Feb. 6, 2013 (JP) ..... 2013-021526

(57) **ABSTRACT**

The present invention is to provide a brush seal device capable of following displacement in a radial direction of a rotational shaft appropriately. The brush seal device prevents leakage of process fluid, wear of a brush seal etc., and leakage of process fluid from an outer circumferential side. The brush seal device **100** of the present invention includes an inner circumferential brush seal **120** and an outer circumferential brush seal **140**. The inner circumferential brush seal **120** has a brush portion **123** and a free end portion **125** at the inner circumferential side. The outer circumferential brush seal **140** is placed at an outer circumferential side of the inner circumferential brush seal **120**, and has a brush portion **143** and a free end portion **145** at the inner circumferential side.

(51) **Int. Cl.**

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**F16J 15/32** (2006.01)

**F01D 11/00** (2006.01)

**F02C 7/28** (2006.01)

(52) **U.S. Cl.**

CPC ..... **F16J 15/3288** (2013.01); **F01D 11/00** (2013.01); **F02C 7/28** (2013.01); **F16J 15/3268** (2013.01)

(58) **Field of Classification Search**

CPC ..... F16J 15/3284; F16J 15/3288

See application file for complete search history.

**8 Claims, 13 Drawing Sheets**

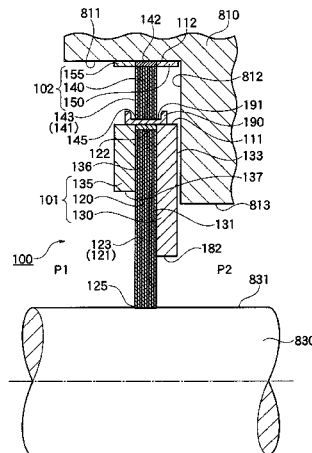


FIG. 1

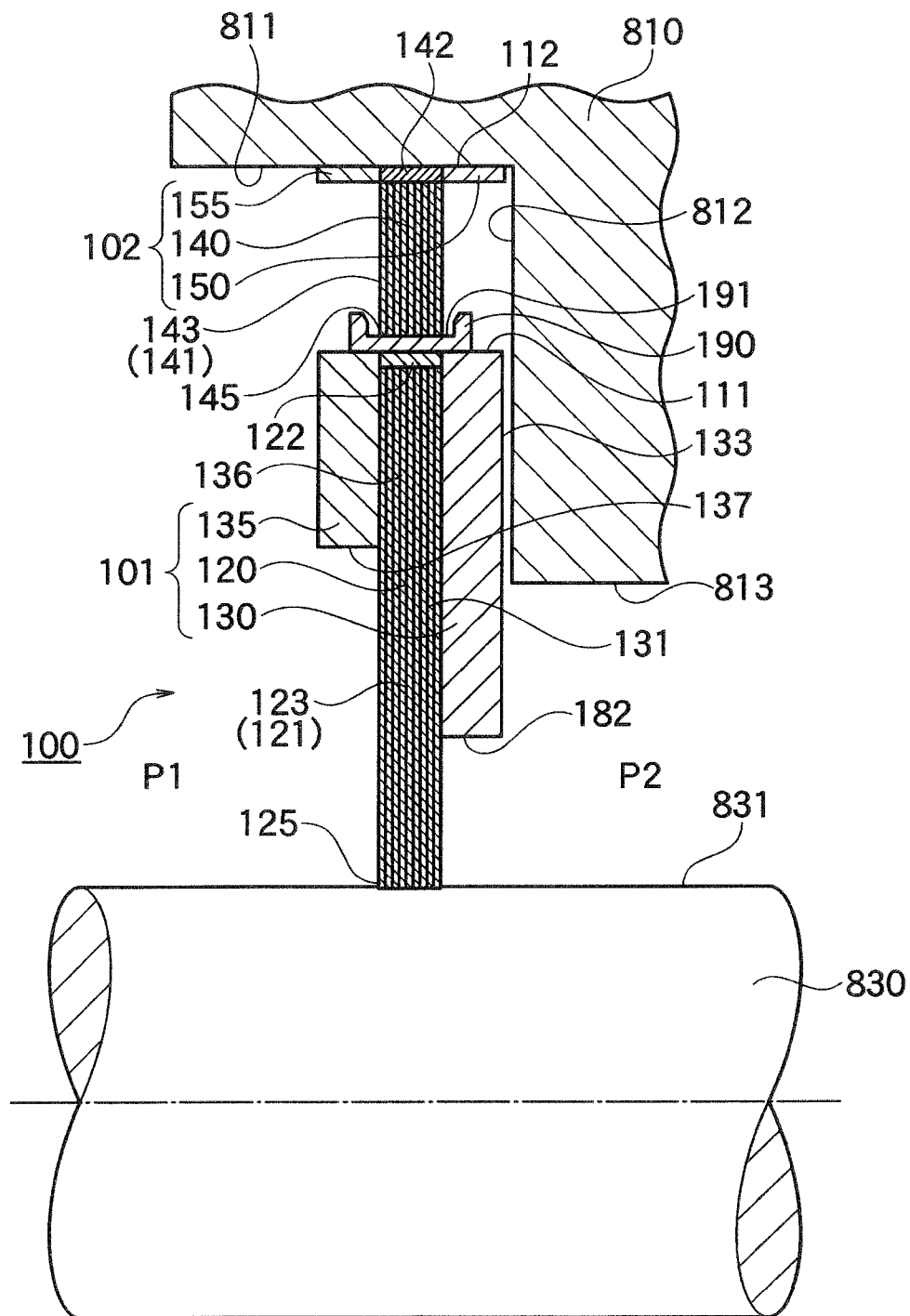


FIG.2A

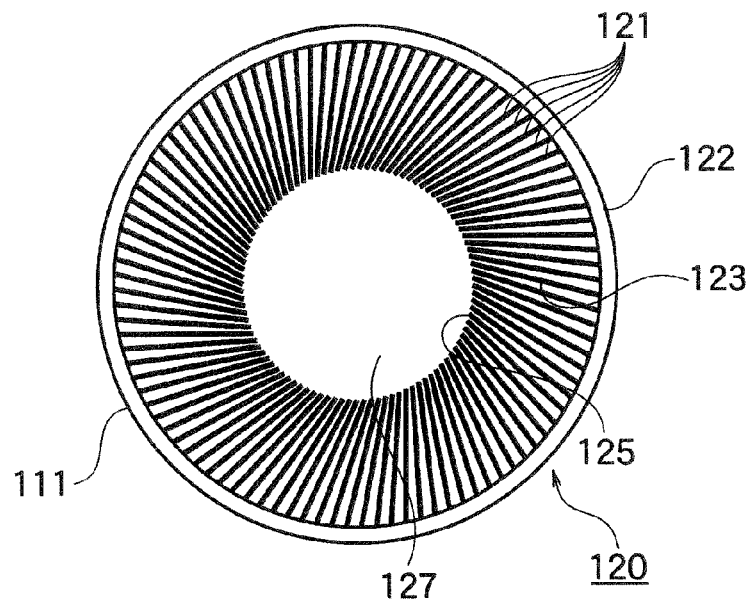


FIG.2B

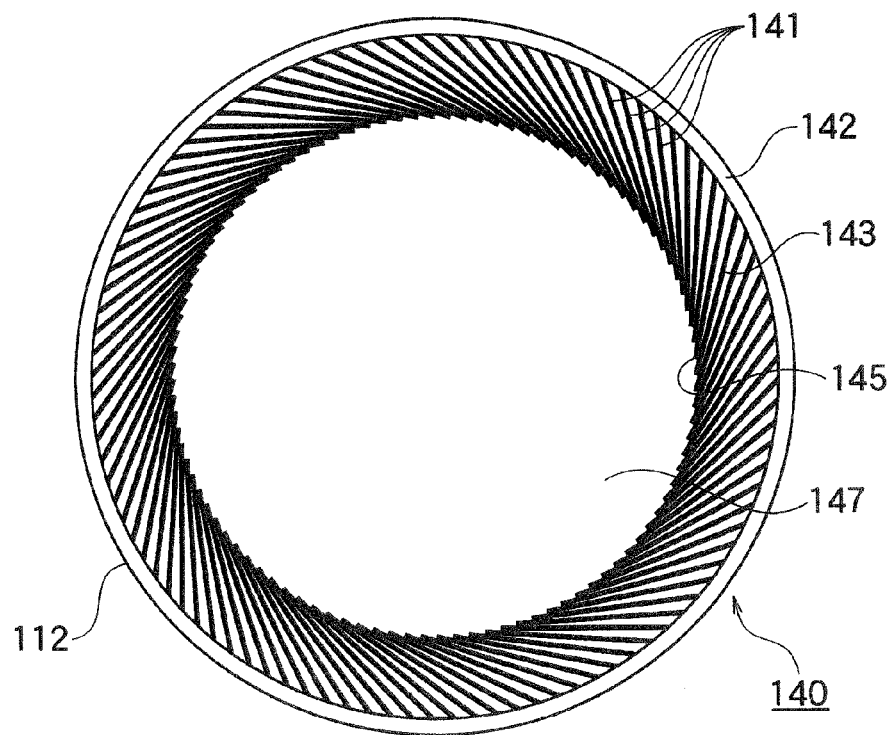


FIG.3

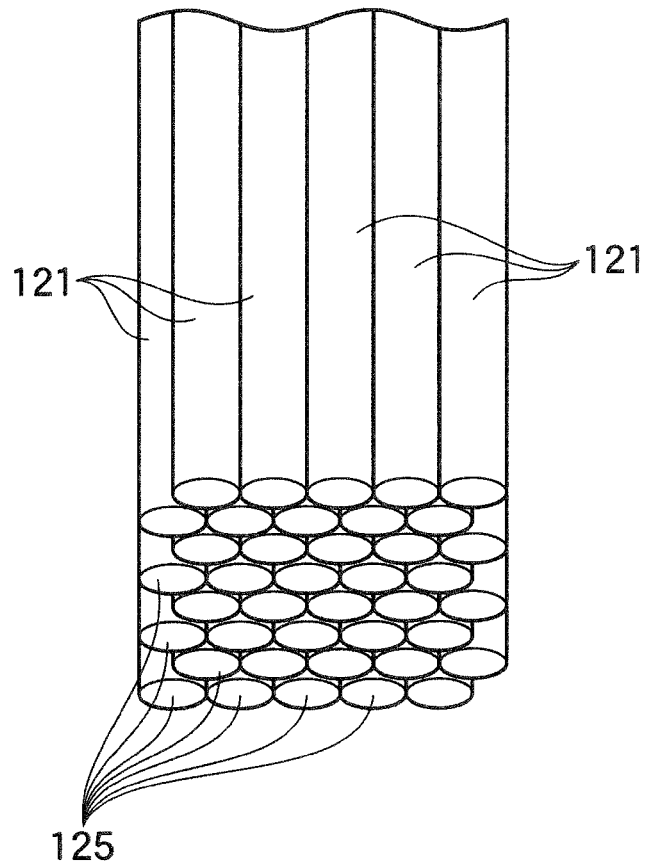


FIG.4

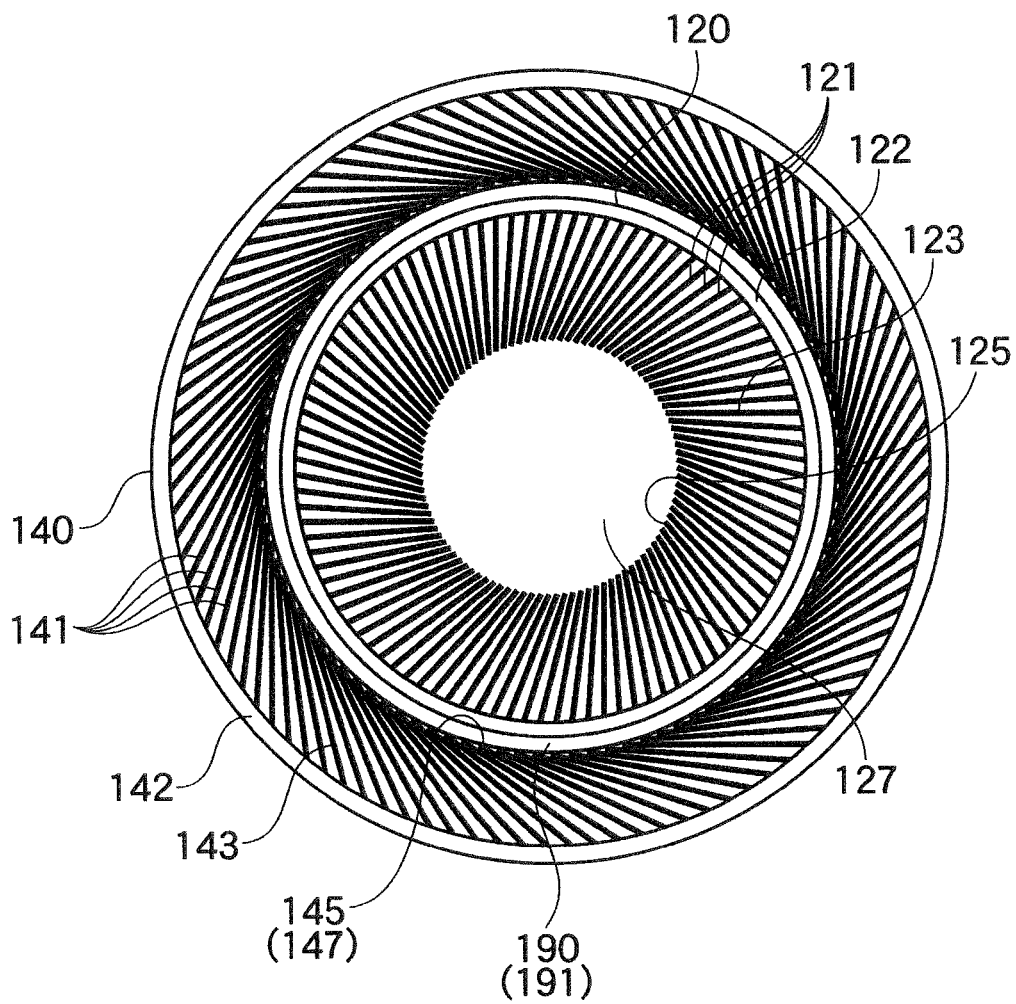


FIG. 5

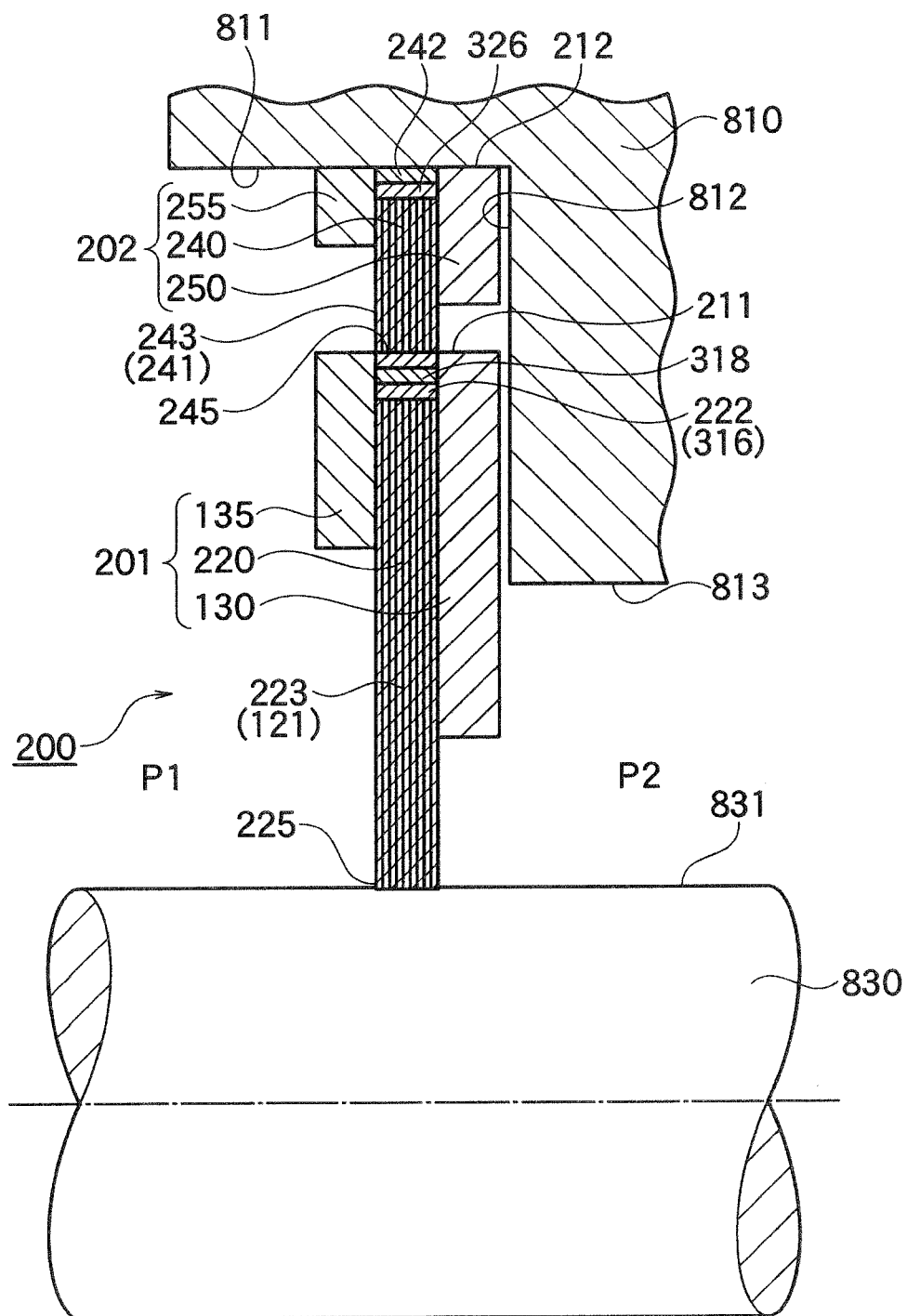


FIG.6A

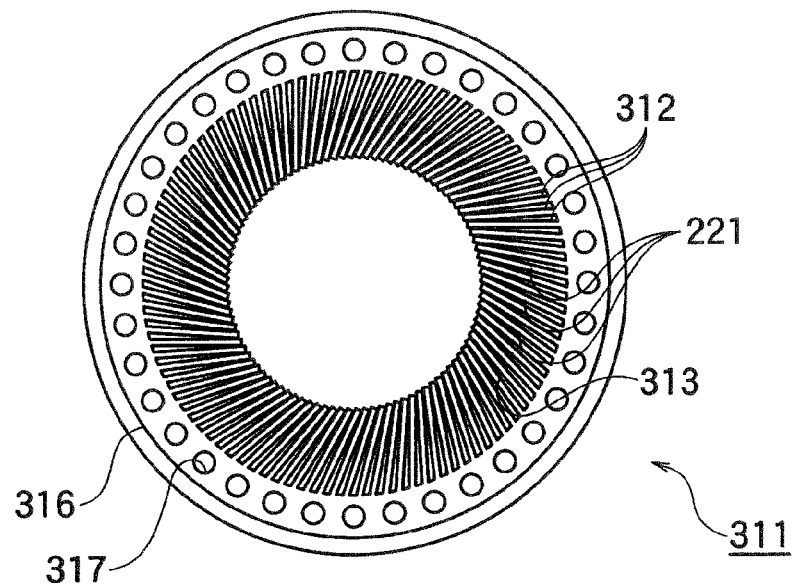


FIG.6B

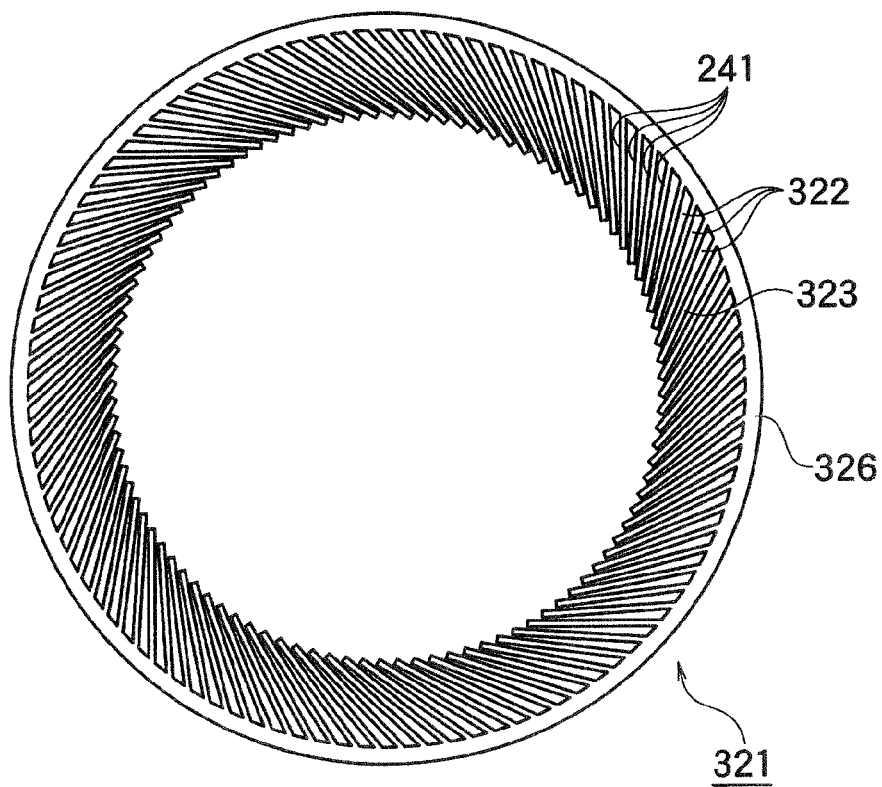


FIG. 7

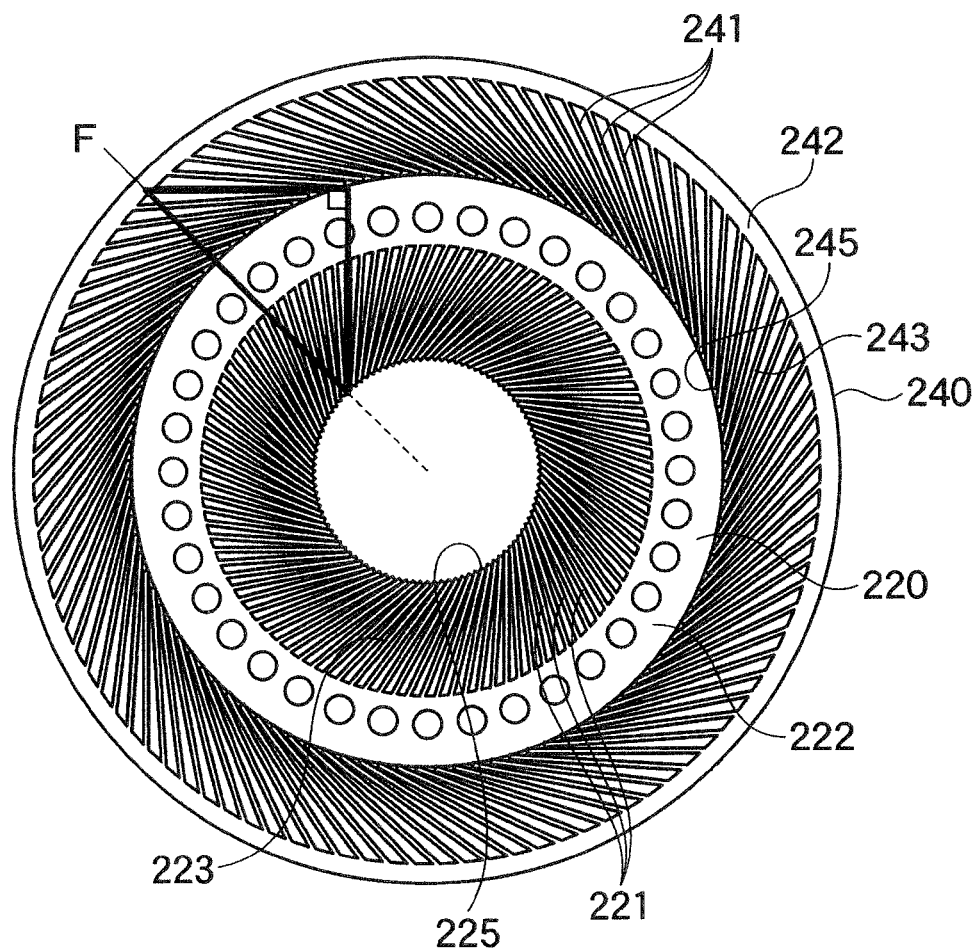




FIG. 8A

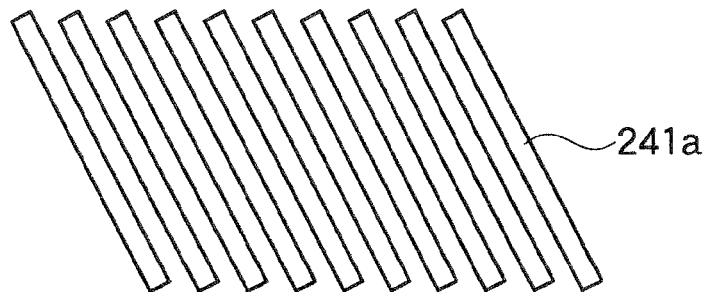


FIG. 8B

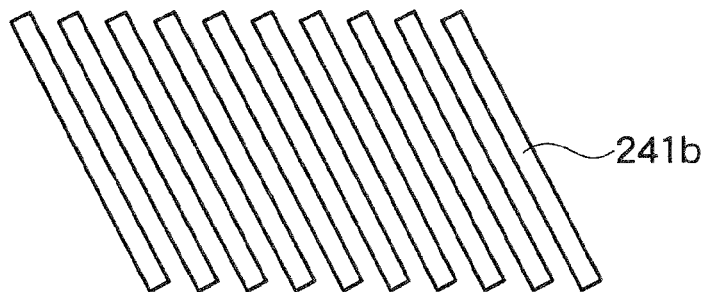


FIG. 8C

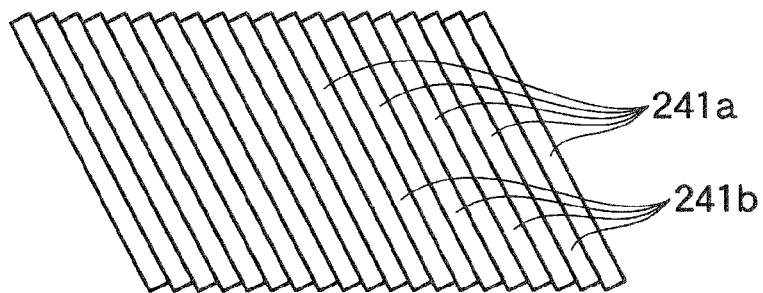


FIG.9A

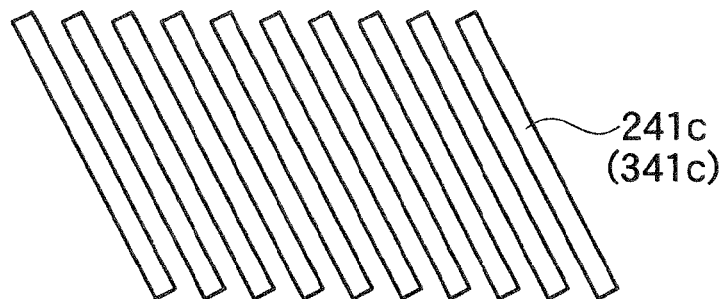


FIG.9B

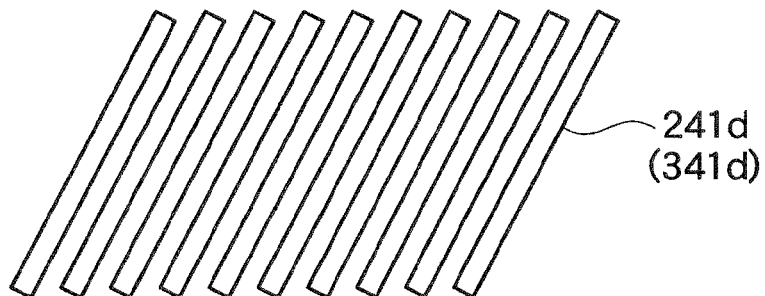


FIG.9C

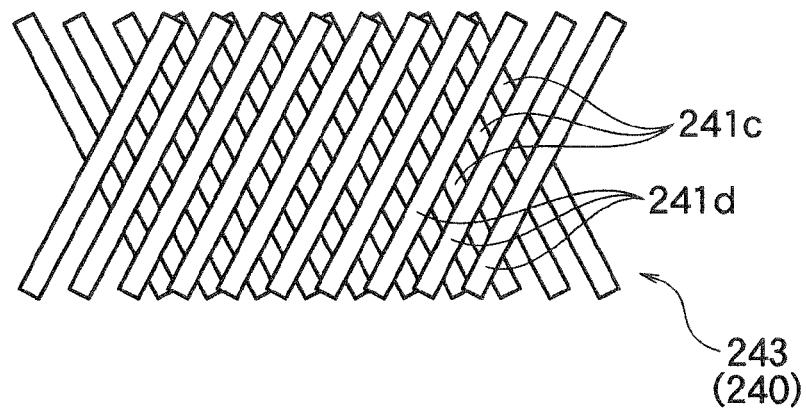


FIG.10A

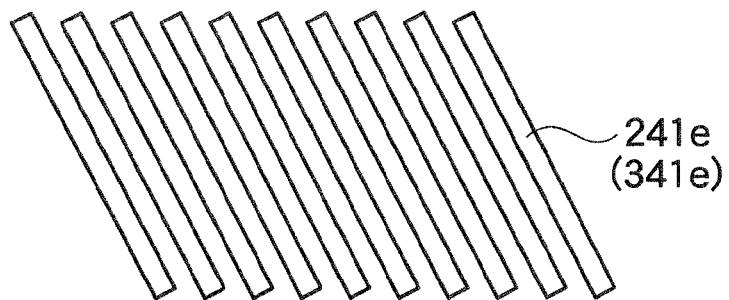


FIG.10B

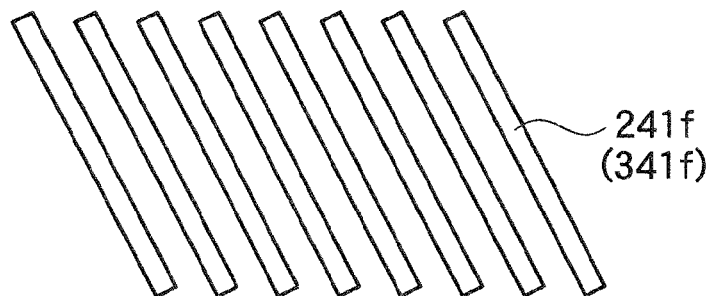


FIG.10C

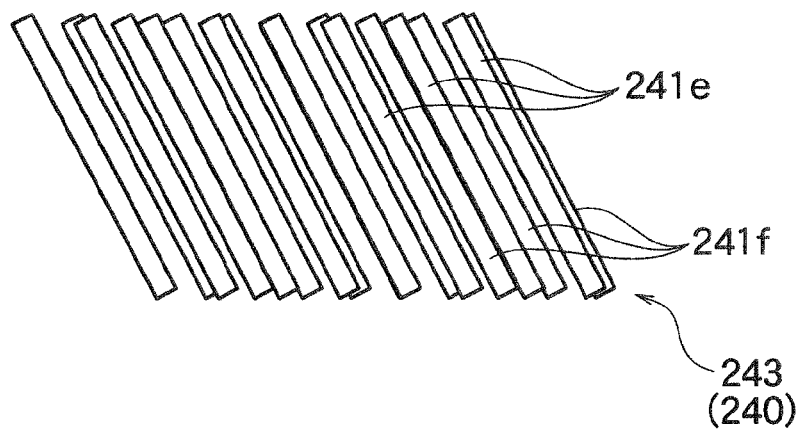


FIG.11A

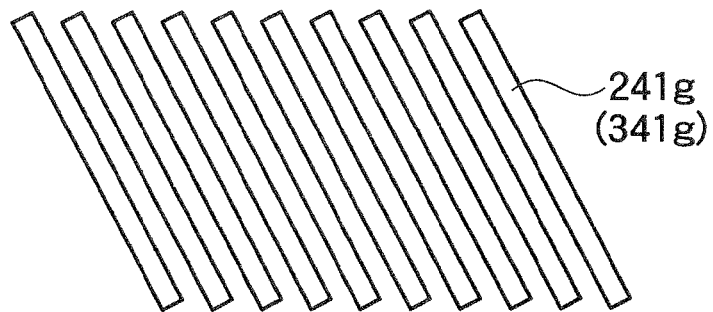


FIG.11B

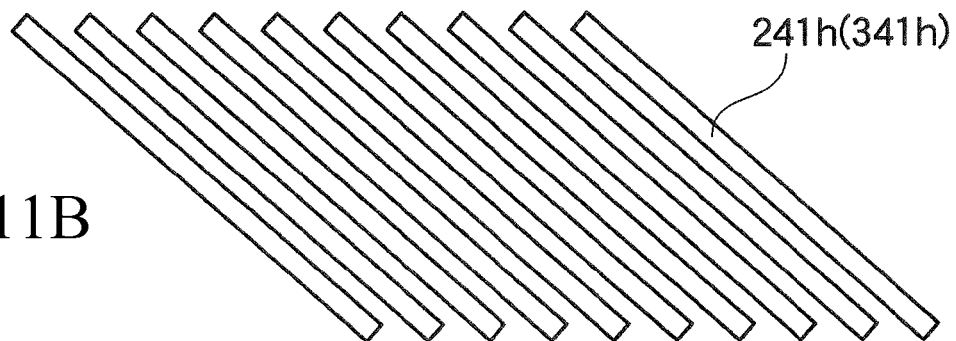


FIG.11C

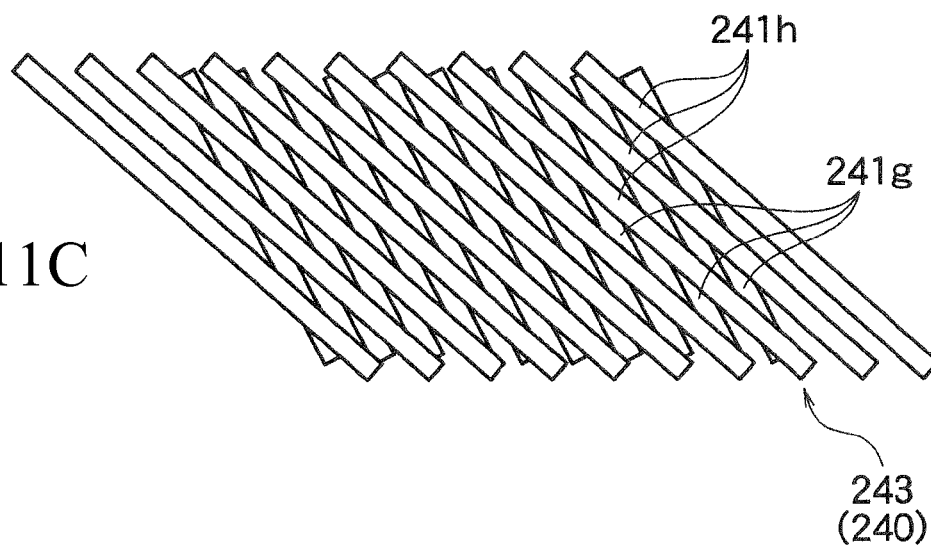


FIG.12

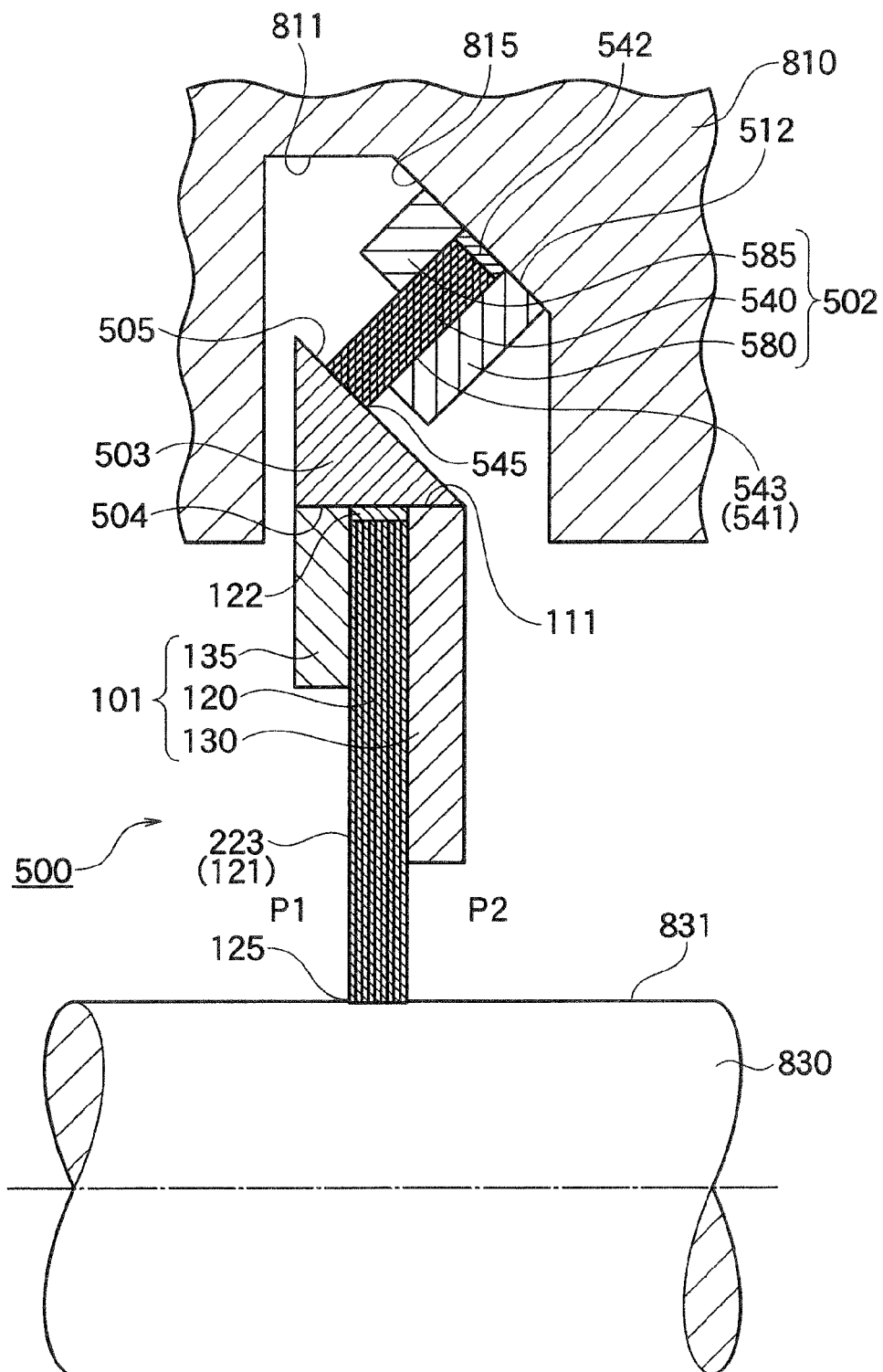


FIG.13A

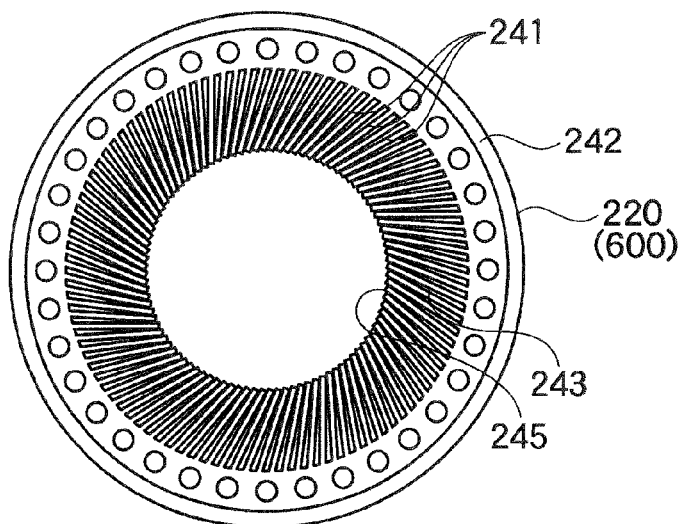
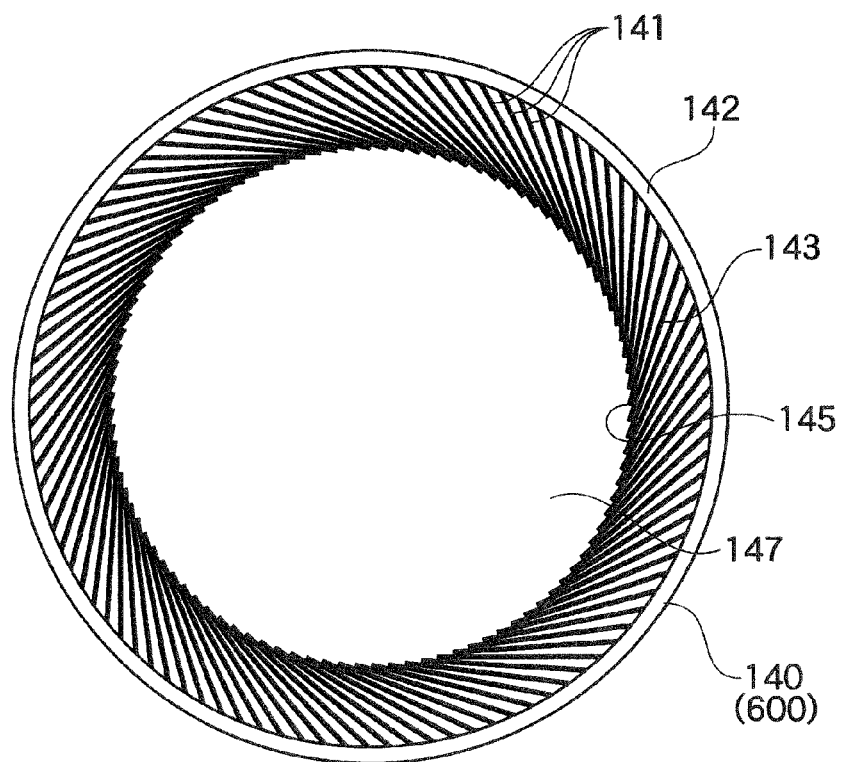


FIG.13B



**BRUSH SEAL DEVICE****CROSS-REFERENCE TO RELATED APPLICATIONS**

This application is a 371 U.S. National Stage of International Application No. PCT/JP2014/052779, filed Feb. 6, 2014, which claims priority to Japanese Patent Application No. 2013-021526, filed Feb. 6, 2013. The entire disclosures of the above applications are incorporated herein by reference.

**TECHNICAL FIELD**

The present invention relates to a brush seal device which seals a clearance between a rotational shaft of such as gas turbines and a mating component such as a housing relatively moving to the rotational shaft. The brush seal device includes a brush portion where a plurality of bristles are arranged.

**BACKGROUND ART**

Brush seal devices which have a brush portion made of multiple bristles are effective because the brush portion is deformed in accordance with movement of a rotational shaft and free end portions of tips of the bristles can follow a circumferential surface of the rotational shaft even if the rotational shaft is slightly moved in a radial direction.

However, when the bristles of a brush seal are pressed by movement of the rotational shaft, it is extremely difficult to control friction force applied thereto in an optimum state. This is because the friction force is affected by spring constant for elastic deformation of the bristles, complex friction force resulting from mounting structure of the bristles, and the like.

When the bristles do not properly follow the displacement of the rotational shaft, space occurs between the bristles and the rotational shaft. Then, this leads to leakage of process fluid or to an early wear of the free end portions of the bristles or a circumferential surface of the rotational shaft.

As a brush seal device to overcome such a problem, that is, as a brush seal device having a higher following ability and less wear, one having a floating-type brush seal body to enable a relatively large movement in the radial direction is known. A brush seal with a plate spring at an outer circumference, which can prevent eccentricity even if space occurs between a rotor and the brush seal due to wear thereof, is also known (for example, see Japanese Laid-Open Patent Publication No. 2011-169319 (Patent Document 1)).

However, for example, in a structure with a floating-type brush seal body or a structure with a plate spring at an outer circumference as shown in Patent Document 1, the outer circumference is not sufficiently sealed. Thus, a secondary seal surface is needed to prevent leakage from the outer circumference. However, in this structure, the outer circumferential side and a portion of the secondary seal are directly exposed to the outer circumferential leakage, which then increases the cost for a sealing ability. In order to prevent the leakage of the secondary seal surface, a high-performance secondary seal is needed to be placed. Thus, in this respect, the cost is also increased. Further, the following ability for the rotational shaft is desired to be improved greatly.

**PRIOR ART**

Patent Document

Patent document 1: Japanese Laid-Open Patent Publication No. 2011-169319

**SUMMARY OF THE INVENTION****Problems to be Solved by the Invention**

The present invention has been achieved to solve such a disadvantage, and its object is to provide a brush seal device achieving the following advantages: The brush seal device can appropriately follow vibration, movement in a radial direction of eccentricity of a rotational shaft or the like with a proper pressure not to be excessive, and as a result, it is possible to prevent leakage of process fluid resulting from gaps between the rotational shaft and a brush seal and to prevent the wear of the brush seal, the rotational shaft, a housing or the like; the brush seal device can appropriately prevent or suppress the leakage from an outer circumferential side without a high-performance secondary seal; and the brush seal device can further appropriately maintain function and effect thereof for a long time.

**Solution to Problem**

In order to solve the above-mentioned problems, the brush seal device of the present invention has a brush portion in which a plurality of bristles are arranged and sealing in an axial direction between a housing and a rotational shaft, the brush seal device comprising:

an inner circumferential brush seal part having an inner circumferential brush seal; and

an outer circumferential brush seal part having an outer circumferential brush seal, wherein

said inner circumferential brush seal comprises:

an attachment section provided at an outer circumferential side;

a brush portion in which a plurality of bristles are arranged from the attachment section to an inner circumferential direction; and

a free end portion which is an inner-circumferential-side end portion of the brush portion and is arranged opposite to an outer circumferential surface of the rotational shaft so as to contact or be adjacent to the outer circumferential surface, and

said outer circumferential brush seal comprises:

an attachment section provided at an outer circumferential side;

a brush portion in which a plurality of bristles are arranged from the attachment section to an inner circumferential direction; and

a free end portion which is an inner-circumferential-side end portion of the brush portion and is arranged opposite to an outer circumferential surface of the inner circumferential brush seal part so as to contact or be adjacent to the outer circumferential surface.

According to the present invention with such a structure, the following brush seal device can be provided: The brush seal device can appropriately follow vibration, movement in a radial direction of eccentricity of a rotational shaft or the like with a proper pressure not to be excessive, and as a result, it is possible to prevent leakage of process fluid resulting from gaps between the rotational shaft and a brush seal and to prevent the wear of the brush seal, the rotational shaft, a housing or the like; the brush seal device can appropriately prevent or suppress the leakage from an outer circumferential side without a high-performance secondary seal; and the brush seal device can further appropriately maintain function and effect thereof for a long time.

Preferably, at least one of said inner circumferential brush seal and said outer circumferential brush seal is a brush seal

3

having a seal part and the free end portion formed on an inner circumferential side of a plurality of wire members bundled and arranged in a wall shape along a circumference and formed into a ring shape by welding an outer end portion of the wire members combined to one another.

Also, preferably, at least one of said inner circumferential brush seal and said outer circumferential brush seal comprises a plurality of plate brush seal unit discs integrally formed by axially laminating themselves,

the plate brush seal unit disc comprises:

the attachment section;

the brush portion having narrow beams as the bristles extending from the attachment section to an inner circumferential side and inclined with respect to a circumferential direction; and

the free end portion.

Also, preferably, the brush portion of the inner circumferential brush seal and the brush portion of the outer circumferential brush seal are composed of bristles of the same material.

Also, preferably, the brush portion of the inner circumferential brush seal and the brush portion of the outer circumferential brush seal are composed of bristles of different materials.

Also, preferably, a plurality of the bristles of the inner-circumferential-side brush portion of the inner circumferential brush seal are arranged toward an inner circumferential direction to be inclined with respect to a radial direction,

a plurality of the bristles of the outer-circumferential-side brush portion of the outer circumferential brush seal are arranged toward an inner circumferential direction to be inclined with respect to a radial direction, and

the bristles of the inner circumferential brush seal and the bristles of the outer circumferential brush seal are inclined oppositely with respect to the radial direction.

Also, preferably, the brush seal device further comprises a brush receiving part placed on the outer circumferential surface of the inner circumferential brush seal part so as to contact or be adjacent to the free end portion of the outer circumferential brush seal and to restrict a spreading out in an axial direction of the free end portion of the outer circumferential brush seal.

Also, preferably, the outer circumferential brush seal is formed to be inclined with respect to the axial direction of the rotational shaft.

#### BRIEF EXPLANATION OF THE DRAWINGS

FIG. 1 is a cross sectional view showing a structure of a brush seal device of the first embodiment of the present invention.

FIG. 2A is a plane view of an inner circumferential brush seal providing an inner circumferential brush seal part of the brush seal device shown in FIG. 1.

FIG. 2B is a plane view of an outer circumferential brush seal providing an outer circumferential brush seal part of the brush seal device shown in FIG. 1.

FIG. 3 is a perspective view showing a form of free end portions of an inner circumferential brush seal of the inner circumferential brush seal part of the brush seal device shown in FIG. 1.

FIG. 4 is a radial-directional cross sectional view of the inner and outer circumferential brush seals of the two-stage inner and outer circumferential brush seal parts of the brush seal device shown in FIG. 1.

4

FIG. 5 is a cross sectional view showing a structure of a brush seal device of the second embodiment of the present invention.

FIG. 6A is a first figure showing a plate brush seal unit disc consisting the brush seal of the brush seal part of the brush seal device shown in FIG. 5, and is a plane view of a plate brush seal unit disc consisting the inner circumferential brush seal of the inner circumferential brush seal part.

FIG. 6B is a second figure showing a plate brush seal unit disc consisting the brush seal of the brush seal part of the brush seal device shown in FIG. 5, and is a plane view of a plate brush seal unit disc consisting the outer circumferential brush seal of the outer circumferential brush seal part.

FIG. 7 is a radial-directional cross sectional view of the inner and outer circumferential brush seals of the two-stage inner and outer circumferential brush seal parts of the brush seal device shown in FIG. 5.

FIG. 8A is a first figure to explain a method for overlapping narrow beams at an inner circumferential brush seal shown in FIG. 6A, and the first figure shows a pattern of first narrow beams.

FIG. 8B is a second figure to explain a method for overlapping narrow beams at an inner circumferential brush seal shown in FIG. 6A, and the second figure shows a pattern of second narrow beams.

FIG. 8C is a third figure to explain a method for overlapping narrow beams at an inner circumferential brush seal shown in FIG. 6A, and the third figure shows that the first narrow beams shown in FIG. 8A and the second narrow beams shown in FIG. 8B are laminated.

FIG. 9A is a first figure showing a first example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the first figure shows a pattern of first narrow beams.

FIG. 9B is a second figure showing a first example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the second figure shows a pattern of second narrow beams.

FIG. 9C is a third figure showing a first example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the third figure shows that the first narrow beams shown in FIG. 9A and the second narrow beams shown in FIG. 9B are laminated.

FIG. 10A is a first figure showing a second example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the first figure shows a pattern of first narrow beams.

FIG. 10B is a second figure showing a second example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the second figure shows a pattern of second narrow beams.

FIG. 10C is a third figure showing a second example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the third figure shows that the first narrow beams shown in FIG. 10A and the second narrow beams shown in FIG. 10B are laminated.

FIG. 11A is a first figure showing a third example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the first figure shows a pattern of first narrow beams.



5

FIG. 11B is a second figure showing a third example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the second figure shows a pattern of second narrow beams.

FIG. 11C is a third figure showing a third example of overlapping of narrow beams at an outer circumferential brush seal as a brush seal device of the third embodiment of the present invention, and the third figure shows that the first narrow beams shown in FIG. 11A and the second narrow beams shown in FIG. 11B are laminated.

FIG. 12 is a cross sectional view showing a structure of a brush seal device of the fourth embodiment of the present invention.

FIG. 13A is a first figure showing a structure of a brush seal device of the fifth embodiment of the present invention, and the first figure shows an inner circumferential brush seal of an inner circumferential brush seal part.

FIG. 13B is a second figure showing a structure of a brush seal device of the fifth embodiment of the present invention, and the second figure shows an outer circumferential brush seal of an outer circumferential brush seal part.

## MODES FOR CARRYING OUT THE INVENTION

### First Embodiment

A brush seal device of the first embodiment of the present invention will be explained with reference to FIG. 1 to FIG. 4.

FIG. 1 is a cross sectional view showing a structure of a brush seal device 100 of the first embodiment of the present invention. FIG. 2A is a plane view of an inner circumferential brush seal 120 of an inner circumferential brush seal part 101 of the brush seal device 100. FIG. 2B is a plane view of an outer circumferential brush seal 140 of an outer circumferential brush seal part 102 of the brush seal device 100. FIG. 3 is a perspective view showing a form of free end portions 125 of the inner circumferential brush seal 120. FIG. 4 is a radial-directional cross sectional view of the inner and outer circumferential brush seals 120 and 140 of the brush seal device 100.

The brush seal device 100 is one used to be arranged in a gas turbine, for example. As shown in FIG. 1, the brush seal device 100 is placed in a gap between a housing 810 and a rotational shaft 830 and divides the gap into a higher pressure side P1 and a lower pressure side P2. The brush seal device 100 is attached to a step shoulder 812 provided on an inner circumferential surface 811 of the housing 810.

The brush seal device 100 has an inner circumferential brush seal part 101 and an outer circumferential brush seal part 102. The inner circumferential brush seal part 101 and the outer circumferential brush seal part 102 have brush portions 123 and 143 at the inner circumferential side, and the tips (the inner-circumferential-side end portions) are formed at the free end portions 125 and 145, respectively.

In the inner circumferential brush seal part 101, the free end portions 125 are arranged opposite to an outer circumferential surface 831 of the rotational shaft 830 so as to contact or be adjacent to the outer circumferential surface 831. In the outer circumferential brush seal part 102, the free end portions 145 are arranged opposite to an outer circumferential surface 111 of the inner circumferential brush seal part 101 so as to contact or be adjacent to the outer circumferential surface 111 at the outer circumferential side of the inner circumferential brush seal part 101. An outer circumferential surface 112 of the outer circumferential brush seal part 102 is closely fitted with the inner circumferential surface 811 of the housing 810.

6

In the brush seal device 100, the two-stage brush seal of the inner circumferential brush seal part 101 and the outer circumferential brush seal part 102 axially seals process fluid at the higher pressure side P1 in a space between the housing 810 and the rotational shaft 830.

The inner circumferential brush seal part 101 has the inner circumferential brush seals 120, a back plate part 130, and a holding part 135.

As shown in FIG. 1 and FIG. 2A, the inner circumferential brush seal 120 is formed into a ring shape by bundling multiple linear bristles 121, arranging the bundled bristles 121 in a wall shape along a circumference, and bonding an outer circumferential side with welding. A ring-shaped welding part at the outer circumferential side is formed as an attachment section 122. An inner circumferential side thereof is formed as a brush portion 123. End portions at the inner circumferential side are formed as the free end portions 125, which contact or are adjacent to the outer circumferential surface 831 of the rotational shaft 830.

As shown in FIG. 2A, each of the bristles 121 has a linear shape and is arranged to be inclined toward the rotational direction of the rotational shaft 830 with respect to a radial direction of the rotational shaft 830 at a predetermined angle. The rotational shaft 830 is fitted with a ring space 127 to be inside of the free end portions 125 of the inner circumferential brush seal 120 arranged in such a way. Thus, the inner circumferential brush seal 120 is formed so that an inner diameter of the ring space 127 is approximately the same as an outer diameter of the outer circumferential surface 831 of the rotational shaft 830.

The free end portions 125 of the bristles 121 of the inner circumferential brush seal 120, which form an engagement surface (engagement part) with the rotational shaft 830, are arranged orderly and densely so as not to make a blank space as much as possible and are subjected to a precision finish as shown in FIG. 3.

In the present embodiment, the bristles 121 are made of steel, nickel-base alloy, or ceramic material etc., and are wire members whose cross sections are circular. The diameter of the bristle 121 is 0.5 to 0.005 mm, and preferably 0.20 to 0.02 mm.

Note that, the bristle 121 may have an optional shape on the presumption that function, movement, or effect etc. described in the present specification can be demonstrated. For example, the bristle 121 may be a member having an optional cross sectional form other than a circular shape such as a wire member having a rectangular cross section, or may be a member having a diameter other than the above range. Also, the bristle may be one whose cross sectional shape, cross sectional size (diameter) or the like changes. Further, the bristles 121 may be used by mixing some bristles having such a different shape or form in multiple bristles.

As shown in FIG. 1, the back plate part 130 is an annular plate member placed at the lower pressure side P2 of the inner circumferential brush seal 120. The back plate part 130 is placed so that a side surface 131 is contacted to the bristles 121 of the inner circumferential brush seal 120, and supports the bristles 121 from the lower pressure side P2 to receive pressure of process fluid.

An opening 132, which makes the rotational shaft 830 go through, is formed on the central position of the back plate part 130. The opening 132 has a diameter which is large enough not to be contacted even if the rotational shaft 830 moves in the axial direction due to vibration or eccentricity etc. thereof. On the other hand, from a perspective of supporting the bristles 121 appropriately and preventing leakage of the process fluid, the opening 132 is formed to have a diam-

7

eter which is smaller than that of an inner-side inner circumferential surface **813** of the step shoulder **812** of the housing **810** or an opening **137** of the holding part **135** mentioned later.

Note that, a secondary seal may be formed between the brush seal device **100** and the housing **810** on a side surface **133** at the lower pressure side **P2** of the back plate part **130**. In that case, the side surface **133** is constituted as a secondary seal surface.

The holding part **135** is an annular plate member placed at the higher pressure side **P1** of the inner circumferential brush seal **120**. In the holding part **135**, a side surface **136** is placed to be contacted to the bristles **121** of the inner circumferential brush seal **120**, and an opening **187** for making the rotational shaft **830** go through is formed on the central position thereof. The opening **137** is formed to have a diameter which is sufficiently larger than that of the opening **132** of the back plate part **130** in order to respond to such as deformation of the bristles **121** when the rotational shaft **830** contacts to the inner circumferential brush seal **120**, the contact state is changed, or the like due to vibration, eccentricity or so.

The back plate part **130** and the holding part **135** are integrally formed with the inner circumferential brush seal **120** to hold it therebetween. That is, the back plate part **130**, the attachment section **122** of the inner circumferential brush seal **120**, and the holding part **135** are welded so that the attachment section **122** of the inner circumferential brush seal **120** is positioned at center, and they are combined as in the trinity. Consequently, the inner circumferential brush seal part **101** is constituted as a whole.

Note that, a brush receiving part **190** mentioned below is placed on the outer circumferential surface **111** of the inner circumferential brush seal part **101**, where the back plate part **130**, the inner circumferential brush seal **120**, and the holding part **135** are integrated.

As shown in FIG. 4, the inner circumferential brush seal part **101** is fitted with a ring space **147** through the brush receiving part **190**. The ring space **147** is the inside of the free end portions **145** of the outer circumferential brush seal **140** of the outer circumferential brush seal part **102**. Thus, the outer diameter of the inner circumferential brush seal part **101**, that is, the outer diameter of the inner circumferential brush seal **120**, the back plate part **130**, and the holding part **135** is approximately the same as the inner diameter of the ring space **147** of the outer circumferential brush seal part **102** mentioned below.

The back plate part **130** and the holding part **135** are manufactured by an optional material such as nickel-base alloy, iron, steel, or other nonferrous metals. In terms of the back plate part **130** and the holding part **135**, a preferable material is properly selected by considering such as type, temperature or other conditions of process fluid, or material or structure of the bristles **121** or the attachment section **122** of the inner circumferential brush seal **120**.

As shown in FIG. 1, the outer circumferential brush seal part **102** has the outer circumferential brush seal **140**, a first holding part **150**, a second holding part **155** and the brush receiving part **190**. The outer circumferential brush seal part **102** is basically constituted in the same way as the inner circumferential brush seal part **101** mentioned above. However, they are different in size of each part, inclined direction of the bristles **141**, structure and function of the holding parts **150** and **155**, presence of the brush receiving part **190**, and the like.

As shown in FIG. 1 and FIG. 2B, the outer circumferential brush seal **140** is formed into a ring shape by bundling the multiple linear bristles **141**, arranging the bundled bristles

8

**141** in a wall shape along the circumference, and bonding the inner circumferential side with welding in the same way of the inner circumferential brush seal **120**. A ring-shaped welding part at the outer circumferential side is formed as an attachment section **142**. The inner circumferential side thereof is formed as the brush portion **143**. The ends of the bristles **141** at the inner circumferential side are formed as the free end portions **145**, which contact or are adjacent to the brush receiving part **190** placed on the outer circumferential surface **111** of the inner circumferential brush seal part **101**.

In the outer circumferential brush seal **140**, as shown in FIG. 4, the inner circumferential brush seal part **101** is fitted in the ring space **147**, which is the inside of the free end portions **145**. Thus, the inner diameter of the ring space **147** arranged at the inside of the free end portions **145** of the outer circumferential brush seal **140** is formed to be approximately the same as the outer diameter of the outer circumferential surface **111** of the inner circumferential brush seal part **101**.

The bristles **141** have substantially the same material, cross sectional shape, cross sectional size, arrangement density and arrangement form as those of the bristles **121** of the inner circumferential brush seal **120** mentioned above.

Note that, in the outer circumferential brush seal **140**, the bristles **141** are inclined in an opposite direction to a direction where the bristles **121** of the inner circumferential brush seal **120** are inclined along the circumferential direction with respect to the radial direction of the rotational shaft **830**, that is, the radial direction of the inner circumferential brush seal part **101** or the outer circumferential brush seal part **102**. Note that, function and effect of such a structure will be mentioned later.

As shown in FIG. 1, the first holding part **150** is an annular member placed at the lower pressure side **P2** of the attachment section **142** of the outer circumferential brush seal **140**. The second holding part **155** is an annular member placed at the higher pressure side **P1** of the attachment section **142** of the outer circumferential brush seal **140**. The first holding part **150** and the second holding part **155** are integrally formed with the outer circumferential brush seal **140** by holding it therebetween. That is, the first holding part **150**, the attachment section **142** of the outer circumferential brush seal **140**, and the second holding part **155** are welded so that the attachment section **142** of the outer circumferential brush seal **140** is positioned at center, and they are combined as in the trinity. As a result, the main part of the outer circumferential brush seal part **102** is constituted as a whole.

When the outer circumferential brush seal part **102** is placed on the housing **810**, in other words, when the brush seal device **100** is placed on the housing **810**, the outer circumferential surface **112** of the integrated first holding part **150**, outer circumferential brush seal **140** and second holding part **155** is formed as an engagement surface closely engaged with the inner circumferential surface **811** of the housing **810**. Thus, the outer diameter of the outer circumferential brush seal part **102**, that is, the outer diameter of the outer circumferential brush seal **140**, the first holding part **150** and the second holding part **155** is formed to be approximately the same as the inner diameter of the inner circumferential surface **811** of the housing **810**.

The brush receiving part **190** is a cylindrical member engaged and placed on the outer circumferential surface of the inner circumferential brush seal part **101**, and is a spreading restriction member for restricting spreading of the free end portions **145** of the bristles **141** of the outer circumferential brush seal **140**. A concave **191** is formed on the brush receiving part **190** toward the outer diameter direction, that is, on the outer circumferential surface thereof. The concave **191**

is a passage of the bristles **141** of the outer circumferential brush seal part **102** so that the free end portions **145** of the outer circumferential brush seal **140** are arranged to be close to, contact to, or slide to the concave **191**. Both end portions in the axial direction of the concave **191** annularly protrude in the outer diameter direction, and restrict the movement of the free end portions **145** of the outer circumferential brush seal **140** so that they do not protrude to the outside of the concave **191**.

The above-mentioned brush receiving part **190** is placed against the free end portions **145** of the outer circumferential brush seal **140**, so that it is possible to restrict spreading of the free end portions **145** of the bristles **141** of the outer circumferential brush seal **140** and to enable the free end portions **145** to be arranged orderly and densely for a long time. In particular, as the present embodiment shown in FIG. 1, the outer circumferential brush seal part **102** with no member pressing side surface of the bristles **141** like the back plate part **130** and the holding part **135** of the inner circumferential brush seal part **101** has a possibility that the free end portions **145** of the outer circumferential brush seal **140** are spread (especially, to the radial direction). However, the brush receiving part **190** is provided, so that the free end portions **145** can be prevented from spreading. As a result, it is possible to prevent lowering of eccentricity prevention function of the brush seal device **100** caused by spreading of the free end portions **145** of the outer circumferential brush seal **140** and to enable the brush seal device **100** to be properly operated for a long time.

Also, with the brush receiving part **190**, it is possible to prevent wear of the outer circumferential surface **111** of the inner circumferential brush seal part **101** caused by directly sliding the free end portions **145** of the outer circumferential brush seal **140**.

As with the back plate part **130** and the holding part **135** of the inner circumferential brush seal part **101**, the first holding part **150**, the second holding part **155** and the brush receiving part **190** are also made from an optional material such as nickel-base alloy, iron, steel, or other nonferrous metals. A preferable material is properly selected for these members by considering type, temperature or other conditions of process fluid, or by considering material, structure or so of the bristles **141** or the attachment section **142** of the outer circumferential brush seal **140**.

The brush seal device **100** having the inner circumferential brush seal part **101** and the outer circumferential brush seal part **102** with such structures is placed between the inner circumferential surface **811** of the housing **810** and the rotational shaft **830** as a brush seal device having an inner and outer circumferential two-stage brush seal as shown in FIG. 1.

In the brush seal device **100**, the inner circumferential brush seal part **101** and the outer circumferential brush seal part **102** can be integrally formed in advance, and the integrally formed brush seal device **100** can be placed between the housing **810** and the rotational shaft **830**. In that case, a setting member, which is attached between the inner circumferential brush seal part **101** and the outer circumferential brush seal part **102** to integrally hold them, may be used based on necessity until the brush seal device **100** is placed on the inner circumferential surface **811** of the housing **810**. The setting member is detached after the brush seal device **100** is placed on the inner circumferential surface **811** of the housing **810** or at a subsequent actual operation stage, and the inner circumferential brush seal part **101** and the outer circumferential brush seal part **102** can be moved relatively.

Instead, in the brush seal device **100**, the inner circumferential brush seal part **101** and the outer circumferential brush

seal part **102** may be separately prepared before the brush seal device **100** is placed (the brush receiving part **190** is preferably placed on the outer circumferential surface **111** of the inner circumferential brush seal part **101**), and they may be assembled when the brush seal device **100** is placed between the housing **810** and the rotational shaft **830**.

In that case, firstly, the outer circumferential brush seal part **102** may be placed on the inner circumferential surface **811** of the housing **810**, and the inner circumferential brush seal part **101** may be then placed on the outer circumferential surface **831** of the rotational shaft **830** in the ring space **127** of the outer circumferential brush seal part **102**. Alternatively, firstly, the inner circumferential brush seal part **101** may be placed on the outer circumferential surface **831** of the rotational shaft **830**, and thereafter the outer circumferential brush seal part **102** may be placed inside of the inner circumferential surface **811** of the housing **810**.

The brush seal device **100** of the present embodiment with such a structure further provides the outer circumferential brush seal **140** at the outer circumferential side of the inner circumferential brush seal **120**. Thus, it is possible to seal process fluid at the outer circumferential side of the brush seal device **100** and to properly prevent leakage of process fluid from the outer circumferential side of the brush seal device **100**.

As a result, a secondary seal provided for preventing leakage of the process fluid at the outer circumferential side of the brush seal device **100** can be unnecessary.

When using the secondary seal simultaneously, it is also possible to properly prevent leakage of the process fluid from a secondary seal surface, to reduce stress of the secondary seal, and to extend its life. Further, it is possible to prevent enhancement of performance required for the secondary seal and to obtain effect such as reduction in cost of devices.

Also, since the brush seal device **100** of the present embodiment has a two-stage brush seal of the inner circumferential brush seal **120** and the outer circumferential brush seal **140**, it is possible to improve elastic characteristic (spring property) of the whole brush seal device **100**. As a result, even if there is a risk that a space occurs between the rotational shaft **830** and the free end portions **125** of the inner circumferential brush seal **120** due to eccentricity of the rotational shaft **830** or wear of the brush seal (the inner circumferential brush seal **120**) etc., the inner circumferential brush seal **120** can properly keep following to the outer circumferential surface **831** of the rotational shaft **830**. As a result, it is possible to prevent such a space from occurring and to properly prevent leakage of process fluid.

In the brush seal device **100** of the present embodiment, the bristles **141** of the outer circumferential brush seal **140** are also formed to be inclined with respect to the inner circumferential surface **811** of the housing **810** and the axial direction of the rotational shaft **830**, and the brush portion **143** of the outer circumferential brush seal **140** also has eccentricity prevention function based on elastic deformation. As a result, the brush seal device **100** as a whole can more preferably perform eccentricity prevention function for the rotational shaft **830**. That is, it is possible to reduce a risk that a space occurs between the rotational shaft **830** and the free end portions **125** of the inner circumferential brush seal **120** and to more preferably perform alignment of the rotational shaft **830** for the brush seal device **100** as a whole.

In the brush seal device **100** of the present embodiment, it is possible to optionally adjust length and angle of the bristles **121** and **141** for each of the inner circumferential brush seal **120** and the outer circumferential brush seal **140**. Therefore, it is possible to respectively adjust eccentricity prevention

11

function performed at the inner circumferential brush seal **120** and the outer circumferential brush seal **140** to a proper operation state. In this respect, the brush seal device **100** as a whole can more preferably perform eccentricity prevention function for the rotational shaft **830**, and can preferably perform alignment of the rotational shaft **830**.

In the brush seal device **100** of the present embodiment, the bristles **141** of the outer circumferential brush seal **140** and the bristles **121** of the inner circumferential brush seal **120** are inclined in different directions with respect to the radial direction of the rotational shaft **830**. That structure makes it possible to generate a force of the outer circumferential brush seal **140** in a direction opposite to the direction of a force of the inner circumferential brush seal **120** generated by the rotation of the rotational shaft **830**. Thus, each of the inner circumferential brush seal **120** and the outer circumferential brush seal **140** can be made harder to be rotated. As a result, wear due to sliding of the free end portions **125** of the inner circumferential brush seal part **101** and the free end portions **145** of the outer circumferential brush seal part **102** is reduced, and the brush seal device **100** can be properly maintained for a long time.

In the brush seal device **100** of the present embodiment, the bristles **121** and the bristles **141** are formed on the inner circumferential brush seal **120** and the outer circumferential brush seal **140** by joining and welding the outer circumferential side, respectively. Joining and welding the bristles **121** and **141** to provide them as the above make it almost certainly possible to prevent the bristles **121** and **141** from falling off and to form the strong brush portions **123** and **143**. As a result, the brush seal device **100** can be properly maintained for a long time.

Note that, in the embodiment mentioned above, the inner circumferential brush seal **120** of the inner circumferential brush seal part **101** and the outer circumferential brush seal **140** of the outer circumferential brush seal part **102** have the same brush portions **123** and **143**. However, the inner circumferential brush seal **120** and the outer circumferential brush seal **140** may have the different brush portions **123** and **143**. That is, the inner circumferential brush seal **120** and the outer circumferential brush seal **140** may constitute the brush portions **123** and **143** with the bristles **121** and **141**, which have different material, diameter, shape (cross sectional shape), characteristic, arrangement density, arrangement form, or the like.

By having such a structure, as the inner circumferential brush seal **120** and the outer circumferential brush seal **140**, it is possible to form the brush portions **123** and **143** having various movement conditions and movement characteristics and to form the brush portions **123** and **143** having optimal conditions respectively. As a result, it is possible to provide a brush seal device which is applicable to a wider technical field and performs a higher performance.

In the present embodiment, a ring member whose cross section is concave is used as the brush receiving part **190**. However, parts defining a range of the outer-circumferential-side free end portions **145** of the outer circumferential brush seal **140** may be just placed at both side of the range in the axial direction on the outer circumferential surface **111** of the inner circumferential brush seal part **101**. As the brush receiving part **190**, an optional one may be used as long as it can restrict spreading of the outer-circumferential-side free end portions **145** of the outer circumferential brush seal **140**.

Accordingly, according to the first embodiment of the present invention, the following brush seal device **100** is provided: Leakage of process fluid from around a rotational shaft and from an outer circumferential side can be prevented;

12

wear of a brush seal, the rotational shaft, or a housing etc. can be prevented; and these function and effect can be properly maintained for a long time.

## Second Embodiment

A brush seal device according to the second embodiment of the present invention will be explained in reference to FIG. **5** to FIG. **8**.

FIG. **5** is a cross sectional view showing a structure of a brush seal device **200** of the second embodiment of the present invention. FIG. **6A** is a plane view of a plate brush seal unit disc **311** consisting an inner circumferential brush seal **220** of an inner circumferential brush seal part **201** of the brush seal device **200**. FIG. **6B** is a plane view of a plate brush seal unit disc **321** consisting an outer circumferential brush seal **240** of an outer circumferential brush seal part **202** of the brush seal device **200**. FIG. **7** is a radial-directional cross sectional view of the inner and outer circumferential brush seals **220** and **240** of the brush seal device **200**. FIG. **8A** and FIG. **8B** are figures showing first and second patterns of narrow beams **221** of the inner circumferential brush seal **220**. FIG. **8C** is a figure showing that they are laminated.

In the following explanation regarding the brush seal device **200** of the second embodiment, the same reference number will be distributed to structures which are substantially the same as those of the brush seal device **100** etc. of the first embodiment mentioned above in reference to FIG. **1** to FIG. **4**, and the structures will not be explained. In the second embodiment, parts which are different from those of the brush seal device **100** of the first embodiment will be mainly explained.

As shown in FIG. **5**, the brush seal device **200** of the second embodiment basically has the same structures as the brush seal device **100** of the first embodiment.

That is, the brush seal device **100** is placed at a clearance between the housing **810** and the rotational shaft **830**, and divides the clearance into the higher pressure side **P1** and the lower pressure side **P2**.

The brush seal device **200** has an inner circumferential brush seal part **201** and an outer circumferential brush seal part **202**. The inner circumferential brush seal part **201** has an inner circumferential brush seal **220**. The outer circumferential brush seal part **202** has an outer circumferential brush seal **240**. The inner circumferential brush seal **220** and the outer circumferential brush seal **240** respectively have brush seal parts **223** and **243** at the inner circumferential side, and tips thereof (inner-circumferential-side end portions) are formed at free end portions **225** and **245**.

In the inner circumferential brush seal part **201**, the free end portions **225** are placed opposite to an outer circumferential surface **831** of the rotational shaft **830** so as to contact or be adjacent to the outer circumferential surface **831**. In the outer circumferential brush seal part **202**, the free end portions **245** are placed opposite to an outer circumferential surface **211** of the inner circumferential brush seal part **201** so as to contact or be adjacent to the outer circumferential surface **211** at the outer circumferential side of the inner circumferential brush seal part **201**. An outer circumferential surface **212** of the outer circumferential brush seal part **202** is closely fitted with the inner circumferential surface **811** of the housing **810**.

With a two-stage brush seal of the inner circumferential brush seal part **201** and the outer circumferential brush seal part **202**, the brush seal device **200** axially seals process fluid at the higher pressure side **P1** in the space between the housing **810** and the rotational shaft **830**.

## 13

First, the brush seal device **200** of the second embodiment is different from the brush seal device **100** of the first embodiment in the structures of the inner circumferential brush seal **220** and the outer circumferential brush seal **240**.

Each of the brush seals **120** and **140** of the inner and outer circumferential brush seal parts **101** and **102** of the above-mentioned brush seal device **100** of the first embodiment is formed in a ring shape by bundling the multiple linear bristles **121** and **141**, arranging the bundled bristles along the circumference in a wall shape, and combining the outer circumferential side, the inner circumferential side, or the middle part with welding.

On the other hand, the inner and outer circumferential brush seals **220** and **240** of the inner and outer circumferential brush seal parts **201** and **202** of the brush seal device **200** of the second embodiment are formed by laminating thin-annular plate brush seal unit discs **311** and **321** as shown in FIG. 6A and FIG. 6B.

The plate brush seal unit discs **311** and **321** respectively have a ring shape. The inner circumferential sides thereof are processed into the narrow beams **221** and **241** inclined with respect to the circumferential direction, and slits **312** and **322** are formed among the narrow beams **221** and **241**. The brush portions **313** and **323** constitute the narrow beams **221**, **241** and the slits **312**, **322** at regular intervals.

Note that, in the plate brush seal unit disc **311** of the inner circumferential brush seal **220**, a direction where the slits **312**, **322** and the narrow beams **221**, **241** are inclined with respect to the circumferential direction is a direction where the rotational shaft **830** rotates. As mentioned below in reference to FIG. 7, an inclined direction, where the narrow beams **221** of the brush seal part **313** of the plate brush seal unit disc **311** are inclined with respect to the radial direction of the rotational shaft **830**, that is, the radial direction of the inner and outer circumferential brush seal parts **201** and **202**, is opposite to that of the plate brush seal unit disc **321** of the outer circumferential brush seal **240**.

The narrow beams **221** and **241** of the plate brush seal unit discs **311** and **321** are bar-shaped (linear, hairy) members having cross sectional shape of rectangle or square whose length of longitudinal and lateral sides is 0.2 to 0.005×0.3 to 0.008 mm, and preferably 0.15 to 0.008×0.2 to 0.01 mm, for example. In addition to rectangle, various shapes such as triangle, trapezoid, or ellipse etc. are used as the cross section of the narrow beams **221** and **241**.

Note that, the narrow beams **221** and **241** may be referred as bristles in general.

In the plate brush seal unit discs **311** and **321**, the outer circumferential sides of the brush portions **313** and **323** are formed into ring plate shaped base portions **316** and **326** without the slits **312**, **322** or the narrow beams **221**, **241**. The narrow beams **221** and **241** are connected to the base portions **316** and **326**. When the plate brush seal unit discs **311** and **321** are laminated to form the inner and outer circumferential brush seals **220** and **240**, the multiple laminated plate brush seal unit discs **311** and **321** are welded to be integrated on the base portions **316** and **317**.

Here, in terms of a method for laminating the plate brush seal unit discs **311** and **321** using the base portions **316** and **326**, an embodiment with different plate brush seal unit discs **311** and **321**, that is, different inner and outer circumferential brush seals **220** and **240** will be exemplified.

In the plate brush seal unit discs **311** constituting the inner circumferential brush seal **220** of the inner circumferential brush seal part **201**, as shown in FIG. 6A, the base portion **316**

## 14

is formed with a relatively wide width, and positioning holes **317** are formed along the circumference on the base portion **316**.

The pitch of the positioning holes **317** does not correspond to that (pitch of the slits **312**) of the narrow beams **221** of the brush portion **313** (the pitch of the positioning holes **317** is not an integral multiple of the pitch of the narrow beams **221**), and is formed to be different slightly. Alternatively, pitch (interval) of the respective positioning holes **317** is formed to be slightly different from each other.

When a plurality of the plate brush seal unit discs **311** are laminated, by shifting and matching the positioning holes **317**, the pitch of the positioning holes **317** is differently formed so that the position of the narrow beams **221** of the brush portion **313** is slightly displaced by the plate brush seal unit discs **311**. Specifically, by properly changing the positioning holes **317** fitted between the plate brush seal unit discs **311**, the position of the narrow beams **221** is displaced by a fraction of the pitches of the narrow beams **221**.

Thus, when the plurality of the plate brush seal unit discs **311** are laminated to form the inner circumferential brush seal **220**, by shifting and matching the positioning holes **317**, it is possible to overlap the narrow beams **221** of other plate brush seal unit discs **311** at the slits **312** of a certain plate brush seal unit disc **311**. This way of laminating the plurality of the plate brush seal unit discs **311** makes it possible to substantially extinguish slits (space) going through the inner circumferential brush seal **220** in the axial direction and to prevent leakage of process fluid effectively, as the whole inner circumferential brush seal **220**.

For example, in the inner circumferential brush seal **220**, by laminating the adjacent plate brush seal unit discs **311** as narrow beams **221a** and **221b** whose pitches are deviated by half as shown in FIG. 8A and FIG. 8B, it is possible to substantially extinguish slits (gaps) going through the inner circumferential brush seal **220** in the axial direction as shown in FIG. 8C.

On the other hand, in the plate brush seal unit discs **321** constituting the outer circumferential brush seal **240** of the outer circumferential brush seal part **202**, as shown in FIG. 6B, the base portion **326** is formed with a relatively small width, and positioning holes or so are not formed on the base portion **326**.

Thus, when the plurality of the plate brush seal unit discs **321** are laminated to form the outer circumferential brush seal **240**, it is impossible to determine the position of plates having the respective outer-circumferential-side slits **322** by using positioning holes like the plate brush seal unit discs **311**. However, in other words, in the outer-circumferential-side slits **322**, each plate can be laminated to be displaced at optional positions. Then, this way of laminating each plate to be displaced makes it possible to substantially extinguish slits (gaps) going through the outer circumferential brush seal **240** in the axial direction and to prevent leakage of process fluid effectively, as the whole outer circumferential brush seal **240**.

That is, in the plate brush seal unit discs **321** without the positioning holes **317**, the base portion **316**, that is, the plate brush seal unit disc **321** itself can be structured more easily than the plate brush seal unit disc **311**. In addition, width of the base portion **326** can be made smaller, and size of the outer circumferential brush seal **240** can be made smaller. Further, the plate brush seal unit discs **321** can be laminated by an optional way of displacement. The plate brush seal unit discs **321** are displaced more freely than the plate brush seal unit discs **311**.

In the present embodiment, the inner circumferential brush seal **220** is consisted of the plate brush seal unit discs **311**

15

having the positioning holes 317, and the outer circumferential brush seal 240 is consisted of the plate brush seal unit discs 321 without the positioning holes 317. However, the inner circumferential brush seal 220 may be consisted of plate brush seal unit discs without positioning holes, and the outer circumferential brush seal 240 may be consisted of plate brush seal unit discs having positioning holes. Both the inner circumferential brush seal 220 and the outer circumferential brush seal 240 may be consisted of plate brush seal unit discs having positioning holes. Both of them may be consisted of plate brush seal unit discs without positioning holes.

The plate brush seal unit discs 311 and 321 to be laminated may be laminated with rotational displacement as mentioned above, or may be laminated without rotational displacement. Such a structure may be employed when leakage amount of process fluid can be sufficiently suppressed even if there is a clearance axially going through the inner circumferential brush seal 220 or the outer circumferential brush seal 240.

A material of the plate brush seal unit discs 311 and 321 is steel, stainless, nickel-base alloy, or ceramic material etc. A conventionally known processing method of a fine machining method such as wire electric discharge machining, etching processing, laser processing, or electron beam processing can be applied to the forming of the slits 312, 322 or the positioning holes 317 etc. for the plate brush seal unit discs 311 and 321.

The inner and outer circumferential brush seals 220 and 240 of the brush seal device 200 of the second embodiment are respectively formed by laminating the plate brush seal unit discs 311 and 321 with such a structure at a necessary laminating thickness or number based on a required seal performance. That is, the plurality of the plate brush seal unit discs 311 and 321 are integrated by positioning the plurality of the plate brush seal unit discs 311 and 321 to be laminated as mentioned above, and by performing such as welding of the base portions 316 and 326.

In the plate brush seal unit disc 311 with the positioning holes 317 on the base portion 316 as shown in FIG. 6A, a plurality of the plate brush seal unit discs 311 can be joined by inserting a positioning pin 318 into the positioning holes 317, which position the plurality of the plate brush seal unit discs 311, as shown in FIG. 5, for example. In this case, when the plurality of the plate brush seal unit discs 311 are welded, the positioning pin 318 may be integrally welded. This is effective for prevention of falling off.

Note that, the positioning pin 318 may be stood on the back plate part 130 in advance, and each of the plate brush seal unit discs 311 may be positioned for the positioning pin 318 placed on the back plate part 130 when the plurality of the plate brush seal unit discs 311 are laminated. In this case, a step for laminating the plurality of the plate brush seal unit discs 311 and a step for placing the back plate part 130 and the holding part 135 are simultaneously performed in substantially the same step. Therefore, the positioning pin 318 may become easy to be handled, and both the step for laminating the plate brush seal unit discs 311 and the step for placing the back plate part 130 and the holding part 135 may become easy steps. The brush seal device 200 may be performed with such a structure and method.

In the plate brush seal unit disc 321 without positioning holes on the base portion 326 as shown in FIG. 6B, a plurality of the plate brush seal unit discs 321 are joined by merely welding the base portion 326 of the respective plate brush seal unit discs 321 while they are maintained to be positioned as desired.

16

Note that, in the inner and outer circumferential brush seals 220 and 240 as formed in this way, welded parts of the base portions 316 and 326 may be referred as attachment sections 222 and 242 hereinafter.

As mentioned above, the brush seal device 200 has the inner and outer circumferential brush seal parts 201 and 202. In addition to the inner circumferential brush seal 220 comprised of the plate brush seal unit discs 311 as mentioned above, the inner circumferential brush seal part 201 has the back plate part 130 and the holding part 135. In addition to the outer circumferential brush seal 240 comprised of the inner-circumferential-side slits 312 as mentioned above, the outer circumferential brush seal part 202 has a back plate part 250 and a holding part 255.

In terms of the inner circumferential brush seal part 201, the back plate part 130 and the holding part 135 are constituted in the same way as the brush seal device 100 of the first embodiment. That is, the back plate part 230, the inner circumferential brush seal 220, and the holding part 135 are welded by centering the attachment section 222 of the inner circumferential brush seal 220 and are combined as in a trinity to form the inner circumferential brush seal part 201.

In terms of the outer circumferential brush seal part 202, structure, function, effect and the like of the back plate part 250 and the holding part 255 are the same as those of the back plate part 130 and the holding part 135 of the inner circumferential brush seal part 201 (the back plate part 130 and the holding part 135 of the inner circumferential brush seal part 101 of the brush seal device 100 of the first embodiment). The back plate part 250, the outer circumferential brush seal 240, and the holding part 255 are welded by centering the attachment section 242 of the outer circumferential brush seal 240 and combined as in a trinity to form the outer circumferential brush seal part 202.

Function, movement, and effect of the free end portions 225, 245 of the respective brush portions 223, 243 of the inner and outer circumferential brush seals 220, 240 of the inner and outer circumferential brush seal parts 201, 202 of the brush seal device 200 are the same as those of the free end portions 125, 145 of the respective brush portions 123, 143 of the brush seals 120, 140 of the respective brush seal parts 101, 102 of the brush seal device 100 of the first embodiment mentioned above.

Note that, in the present embodiment, the brush portions 223, 243 of the inner and outer circumferential brush seals 220, 240 of the inner and outer circumferential brush seal parts 201, 202 are the same brush portions. That is, the narrow beams 221 consisting the inner circumferential brush seal 220 and the narrow beams 241 consisting the outer circumferential brush seal 240 are consisted to have the same material, cross sectional shape, size thereof, characteristic such as elastic characteristic, arrangement density, or arrangement form.

In the brush seal device 100 of the first embodiment, the brush receiving part 190 is placed on the outer circumferential surface 111 of the inner circumferential brush seal part 101, which prevents spreading of the free end portions 145 of the bristles 141 of the outer circumferential brush seal 140 of the outer circumferential brush seal part 102.

On the other hand, in the brush seal device 200 of the second embodiment, the brush receiving part 190 is not provided, and the free end portions 245 of the brush portion 243 of the outer circumferential brush seal 240 are directly contacted to the outer circumferential surface 211 of the inner circumferential brush seal part 201.

In the brush seal device 200, instead of the first and second holding parts 150 and 155 of the brush seal device 100, the back plate part 250 and the holding part 255 hold the outer

17

circumferential brush seal **240** therebetween, and side surfaces of the narrow beams **241** of the outer circumferential brush seal **240** are supported as shown in FIG. 5. That is, in the brush seal device **200**, the back plate part **250** and the holding part **255** hold an area therebetween including root portions of the outer circumferential brush seal **240** near the attachment section **242** by centering the attachment section (base portion) **242** as mentioned above. As a result, in the brush seal device **200**, the back plate part **250** and the holding part **255** prevent spreading of the narrow beams **241** of the outer circumferential brush seal **240**. Thus, it is not necessary to provide a member like the brush receiving part **190** at the part of the free end portions **245** of the outer circumferential brush seal **240**.

Note that, such an embodiment of holding the root portions of the outer circumferential brush seal **240** with the back plate part **250** and the holding part **255** therebetween can be also applied to the bristles **141** of the outer circumferential brush seal **140** of the brush seal device **100** of the first embodiment mentioned above.

FIG. 7 shows a radial-directional cross sectional view at the inner and outer circumferential brush seals **220** and **240** of the brush seal device **200**. As shown in FIG. 7 and as mentioned above, in the brush seal device **200** of the present embodiment, the narrow beams **241** of the outer circumferential brush seal **240** are inclined with respect to a direction opposite to that of the narrow beams **221** of the inner circumferential brush seal **220** inclined with respect to the radial direction of the rotational shaft **830**. In this structure, the tilt angle and length of the narrow beams **221** and **241** are adjusted so that spring effect generated by the narrow beams **241** and **221** of the outer and inner circumferential brush seals **240** and **220** remains only as the radial direction element F of the brush seal device **200**.

In accordance with such a structure, there is only a force in the radial direction with respect to stress (shaking stress) for such as vibration which the brush seal device **200** receives from the rotational shaft **830**. Thus, behavior such as an unstable vibration of the brush seal device **200** can be prevented, and the movement of the brush seal device **200** can be stabilized.

Note that, the features of the tilt angle and length of the narrow beams **221** and **241** can be also applied to a relation of the bristles **121**, **141** of the inner and outer circumferential brush seals **120**, **140** of the brush seal device **100** of the first embodiment mentioned above.

The brush seal device **200** of the second embodiment has such a structure.

Therefore, in the second embodiment, the following brush seal device can be also provided: Leakage of process fluid from around a rotational shaft and from an outer circumferential side can be prevented; wear of a brush seal, the rotational shaft or a housing etc. can be prevented; and these function and effect can be properly maintained for a long time.

Note that, in the present embodiment, the brush portions **223**, **243** of the inner and outer circumferential brush seals **220**, **240** of the inner and outer circumferential brush seal parts **201**, **202** are the same brush portions. However, they can be consisted as different brush portions. Specifically, they become different brush portions by processing the narrow beams **221** constituting the brush seal **220** and the narrow beams **241** constituting the outer circumferential brush seal **240** to have different cross sectional shape, size of cross section, characteristic such as elastic characteristic, arrangement density, or arrangement form etc. As a result, it is pos-

18

sible to provide a brush seal device which is applicable to a wider technical field and performs a higher performance.

In the brush seal device **200**, the inner and outer circumferential brush seal parts **201**, **202** may be integrated in advance before the brush seal device **200** is placed in the housing **810**, or may be separated until the brush seal device **200** is placed in the housing **810** and be integrated when they are placed in the housing **810** (when they are assembled).

### Third Embodiment

A brush seal device of the third embodiment of the present invention will be explained in reference to FIG. 9 to FIG. 12.

The brush seal device of the third embodiment will be explained as a variation of the brush seal device **200** of the second embodiment.

In the brush seal device **200** mentioned above as the second embodiment, the brush seals **220**, **240** (the inner and outer circumferential brush seals **220**, **240**) are constituted by laminating the same plate brush seal unit discs **311**, **321**. However, the brush seals **220**, **240** may be constituted by laminating different types of plate brush seal unit discs.

Especially, the outer circumferential brush seal **240** is formed by laminating various types of the plate brush seal unit discs **321** having different forms of the narrow beams **241**, which can reduce the clearance in the axial direction of the brush portion **243** of the outer circumferential brush seal **240**.

For example, as shown in FIG. 9A and FIG. 9B, two types of the plate brush seal unit discs **321c**, **321d** having narrow beams **241c**, **241d** whose tilt directions are different are laminated, which makes the brush portion **243** of the outer circumferential brush seal **240** have a form as shown in FIG. 9C and makes the clearance in the axial direction smaller.

As shown in FIG. 10A and FIG. 10B, two types of the plate brush seal unit discs **321e**, **321f** having narrow beams **241e**, **241f** whose pitches are different are laminated, which makes the brush portion **243** of the outer circumferential brush seal **240** have a form as shown in FIG. 10C and makes the clearance in the axial direction smaller in this structure as well.

Further, as shown in FIG. 11A and FIG. 11B, two types of the plate brush seal unit discs **321g**, **321h** having narrow beams **241g**, **241h** whose tilt angles are different are laminated, which makes the brush portion **243** of the outer circumferential brush seal **240** have a form as shown in FIG. 11C and makes the clearance in the axial direction smaller in this structure as well.

Since the clearance in the axial direction of the narrow beams **241** of the outer circumferential brush seal **240** becomes small in all the structures, it is possible to reduce or prevent the leakage of process fluid around the outer circumference of the brush seal device **200**, and this is preferable.

The brush seal device of the third embodiment has such a structure.

Function, effect, variation or the like regarding the brush seal device of the respective embodiments mentioned above is also applicable to the brush seal device explained as the third embodiment. Therefore, in the third embodiment, the following brush seal device can be also provided: Leakage of process fluid from around a rotational shaft and from an outer circumferential side can be prevented; wear of a brush seal, the rotational shaft, or a housing etc. can be prevented; and these function and effect can be properly maintained for a long time.

Note that, in order to have different forms of the narrow beams **241**, there are various methods and forms in addition to the above, and an optional method may be applied.

The structure is not limited to a form which narrow beams are formed on plate brush seal unit discs and they are laminated. The structure is also applicable to one forming an outer circumferential brush seal with the bristles explained as the first embodiment.

The same method is also applicable to the inner circumferential brush seal **220**. In that structure, it is possible to reduce or prevent the leakage of process fluid at the brush portion **223** of the inner circumferential brush seal **220**, and this is preferable.

#### Fourth Embodiment

The fourth embodiment of the present invention will be explained in reference to FIG. **12**.

In the following explanation regarding a brush seal device **500** of the fourth embodiment, the same reference number will be distributed to structures which are substantially the same as those of the brush seal devices of each embodiment mentioned above, and the structures will not be explained.

In the brush seal devices of each embodiment mentioned above, the inner circumferential brush seals **120**, **220** and the outer circumferential brush seals **140**, **240** are provided so that a direction of the bristles **121**, **141** or the narrow beams **221**, **241** is along the radial direction of the rotational shaft **830**.

On the other hand, the brush seal device **500** of the fourth embodiment provides an outer circumferential brush seal **540** inclined in the axial direction.

Hereinafter, the brush seal device **500** of the fourth embodiment will be explained in detail in reference to the figure.

As shown in FIG. **12**, the brush seal device **500** of the present embodiment is also one which separates the gap between the housing **810** of such as a gas turbine and the rotational shaft **830** to the higher pressure side **P1** and the lower pressure side **P2**, for example. The brush seal device **500** is attached on an inclined surface **815**, which is provided on the inner circumferential surface **811** of the housing **810**. The inclined surface **815** is a surface inclined with respect to the inner circumferential surface **811** at approximately 45 degrees.

The brush seal device **500** has the inner circumferential brush seal part **101**, an outer circumferential brush seal part **502**, and a connection part **503**.

The inner circumferential brush seal part **101** has the inner circumferential brush seal **120**, the back plate part **130** and the holding part **135**, and has the same structure as the inner circumferential brush seal part **101** of the first embodiment mentioned above.

The outer circumferential brush seal part **502** has the outer circumferential brush seal **540**, a back plate part **580**, and a holding part **585**.

The outer circumferential brush seal **540** of the outer circumferential brush seal part **502** has approximately the same structure as the outer circumferential brush seal **140** of the brush seal device **100** of the first embodiment. However, the entire shape thereof is not a plain annular shape like the outer circumferential brush seal **140** of the first embodiment, but is a truncated conical body (a truncated cone) shape as shown in FIG. **12**. That is, the outer circumferential brush seal **540** is formed into a truncated shape by bundling the multiple linear bristles **541**, arranging the bundled bristles **541** along the truncated cone side circumferential shape in a wall shape, and combining an outer-circumferential-side end portion **542** with welding. Then, the inner circumferential side of the

welding part **542** is formed at a brush portion **543**, and free end portions **545** are formed at the tip thereof.

The outer circumferential brush seal **540** is held by the back plate part **580** and the holding part **585** therebetween and is integrated therewith to obtain the outer circumferential brush seal part **502**.

The back plate part **580** is a member which has the same truncated conical shape (a truncated cone) as the outer circumferential brush seal **540** and is placed at the lower pressure side **P2** of the outer circumferential brush seal **540**. The back plate part **580** supports the bristles **541** of the outer circumferential brush seal **540** from the lower pressure side **P2**.

The holding part **585** is a member which has the same truncated conical shape (a truncated cone) as the outer circumferential brush seal **540** and is placed at the higher pressure side **P1** of the outer circumferential brush seal **540**. The holding part **585** is a member for holding the outer circumferential brush seal **540** with the back plate part **580**.

The back plate part **580** and the holding part **585** are arranged to hold the outer circumferential brush seal **540** therebetween, and are welded by centering the outer-circumferential-side welding part **542** of the outer circumferential brush seal **540**. Then, they are combined as in a trinity and formed into the outer circumferential brush seal part **502**.

The material of the back plate part **580** and the holding part **585** is the same as that of the back plate part **130**, the holding part **135** or the like of the first embodiment.

In each part of the annular-shaped outer circumferential brush seal part **502**, as FIG. **2** shows the cross section, although an outer circumferential surface **512** is formed to be vertical to each plate surface of an outer circumferential brush seal **520**, the back plate part **580** and the holding part **585**, the outer circumferential brush seal part **502** is formed in a truncated conical shape as a whole. Thus, as a whole of the outer circumferential brush seal part **502**, the outer circumferential surface **512** is a surface inclined with respect to the outer circumferential surface **831** of the rotational shaft **830** or the inner circumferential surface **811** of the housing **810**. Then, in the outer circumferential brush seal part **502**, the tilt angle of the outer circumferential surface **512** with respect to the inner circumferential surface **811** of the housing **810** is the same as that of the inclined surface **815** of the housing **810**, and the outer circumferential surface is formed to be opposite to the inclined surface **815** of the housing **810** in parallel.

As shown in FIG. **12**, the connection part **503** is a ring member whose cross section is triangle. The connection part **503** has an inner circumferential surface **504** and an inclined surface **505**, which is parallel to the inclined surface **815** formed on the inner circumferential surface **811** of the housing **810**.

The inner diameter of the connection part **503** is formed to be approximately the same as the outer diameter of the inner circumferential brush seal part **101**. The length (thickness) in the axial direction of the connection part **503** is approximately the same as the length (thickness) in the axial direction of the inner circumferential brush seal part **101**. The inner circumferential brush seal part **101** is fitted and joined with the inner circumferential surface **504** of the connection part **503** as illustrated, and the connection part **503** is integrated with the inner circumferential brush seal part **101**.

As shown in FIG. **12**, the brush seal device **500** with such a structure is placed between the housing **810** and the rotational shaft **830**. In the brush seal device **500**, the outer circumferential surface **512** of the outer circumferential brush seal part **502** is fixed and placed on the inclined surface **815** of the inner circumferential surface **811** of the housing **810**.



## 21

Further, the inclined surface **505** of the connection part **503**, which is placed on the outer circumferential surface **111** of the inner circumferential brush seal part **101** fitted and placed on the rotational shaft **830**, is approximately vertically contacted to the free end portions **545** of the bristles **541** of the outer circumferential brush seal **540** of the outer circumferential brush seal part **502**.

The free end portions **125** of the bristles **121** of the inner circumferential brush seal **120** of the inner circumferential brush seal part **101** are arranged to be adjacent to the outer circumferential surface **831** of the rotational shaft **830**, and seal the clearance near the rotational shaft **830** between the higher pressure side **P1** and the lower pressure side **P2**. Further, free end portions **525** of bristles **521** of the outer circumferential brush seal **540** of the outer circumferential brush seal part **502** are arranged to be adjacent to the inclined surface **505** of the connection part **503**, and seal the clearance near the connection part **503** between the higher pressure side **P1** and the lower pressure side **P2**.

Function, movement, and effect of the brush portions **123**, **543** and the free end portions **125**, **545** of the inner and outer circumferential brush seals **120**, **540** of the inner and outer circumferential brush seal parts **101**, **502** of the brush seal device **500** are the same as those of the brush seal devices of each embodiment mentioned above.

Especially in the brush seal device **500**, the outer circumferential brush seal **540** of the outer circumferential brush seal part **502** is provided to be inclined with respect to the axial direction of the rotational shaft **830**. Thus, the brush seal device **500** can follow not only the movement in the radial direction of the rotational shaft **830**, but also the movement in the axial direction.

The brush seal device **500** of the fourth embodiment has such a structure.

All of function, effect, variation and the like of the brush seal device of the respective embodiments mentioned above are also applicable to the brush seal device **500** of the fourth embodiment. Therefore, in the fourth embodiment, the following brush seal device can be also provided: Leakage of process fluid from around a rotational shaft and from an outer circumferential side can be prevented; wear of a brush seal, the rotational shaft, or a housing etc. can be prevented; and these function and effect can be properly maintained for a long time.

In the brush seal device **500** of the present embodiment, where the outer circumferential brush seal **520** and the outer circumferential brush seal **540** are inclined with respect to the axial direction of the rotational shaft **830**, it is possible to provide the following brush seal device: Displacement in the axial direction of the rotational shaft **830** can be followed; a more stable operation can be performed, and as a result, leakage of process fluid can be more securely prevented; wear of each part can be prevented; and a long term use is possible.

Note that, in the present embodiment, the inner and outer circumferential brush seals **120**, **540** with the bristles **121**, **541** are exemplified, but a brush seal constituted by laminating plate brush seal unit discs as the second embodiment may be used.

In the present embodiment, the outer circumferential brush seal **540** is inclined at 45 degrees, but it may be inclined at an optional degree based on an application target, pressure of process fluid, strength of stress in each direction, vibration, form or size of deviation, or the like.

## Fifth Embodiment

The fifth embodiment of the present invention will be explained in reference to FIG. 13A and FIG. 13B.

## 22

In the following explanation regarding a brush seal device **600** of the fifth embodiment, the same reference number will be also distributed to structures which are substantially the same as those of the brush seal devices of each of the above-mentioned embodiments, and the structures will not be explained, either.

The brush seals of each embodiment mentioned above are constituted by commonly using either an embodiment of bundling and welding bristles or an embodiment of laminating plate brush seal unit discs as the inner and outer circumferential brush seals of the inner and outer circumferential brush seal parts. However, the inner and outer circumferential brush seals may employ different embodiments. In the present embodiment, the brush seal device **600**, which uses both a brush seal with the laminated plate brush seal unit discs as the inner circumferential brush seal and a brush seal with bristles as the outer circumferential brush seal, will be explained.

The brush seal device **600** of the present embodiment has inner and outer circumferential brush seal parts as with each embodiment mentioned above. As shown in FIG. 13A and FIG. 13B, the brush seal device **600** has the inner circumferential brush seal **220** of the second embodiment mentioned above as the inner circumferential brush seal of the inner circumferential brush seal part. Further, the brush seal device **600** has the outer circumferential brush seal **140** of the first embodiment mentioned above as the outer circumferential brush seal of the outer circumferential brush seal part. Both of them have completely the same structures as those of the inner and outer circumferential brush seals **220** and **140** in detail.

Note that, here, only the brush seal will be illustrated, and back plate parts, holding parts and the like will not be illustrated.

The brush seal device **600** is constituted as follows: The inner circumferential brush seal part having the inner circumferential brush seal **220** is placed on the outer circumferential surface **831** of the rotational shaft **830**; the outer circumferential brush seal part having the outer circumferential brush seal **140** is placed on the inner circumferential surface **811** of the housing **810**; and the free end portions **145** of the bristles **143** of the outer circumferential brush seal **140** are arranged to be adjacent to the outer circumferential surface of the inner circumferential brush seal part having the inner circumferential brush seal **220**.

The brush seal device **600** of the fifth embodiment has such a structure.

Function, effect, variation or the like regarding the brush seal device of the respective embodiments mentioned above is also applicable to the brush seal device **600** explained as the fifth embodiment. Therefore, in the fifth embodiment, the following brush seal device can be provided: Leakage of process fluid from around a rotational shaft and from an outer circumferential side can be prevented; wear of a brush seal, the rotational shaft, or a housing etc. can be prevented; these function and effect can be properly maintained for a long time.

Especially, the brush seal device **600** of the present embodiment uses both the brush seal **220**, where plate brush seal unit discs are laminated, as the inner circumferential brush seal and the brush seal **140** formed with bristles as the outer circumferential brush seal. In this way, by combining optional brush seals, it is possible to form brush portions having various movement conditions and movement characteristics and to form brush portions having optimal conditions respectively as the inner and outer circumferential brush

## 23

seals. As a result, it is possible to provide a brush seal device which is applicable to a wider technical field and performs a higher performance.

Note that, the present invention is not limited to the embodiments mentioned above, but can be variously changed within a scope thereof.

## INDUSTRIAL APPLICATION FIELD

The brush seal according to the present invention can be preferably used for an optional device having a rotational shaft of aircrafts, gas turbines and the like.

The invention claimed is:

1. A brush seal device having a brush portion in which a plurality of bristles are arranged and sealing in an axial direction between a housing and a rotational shaft, the brush seal device comprising:

an inner circumferential brush seal part having an inner circumferential brush seal; and

an outer circumferential brush seal part having an outer circumferential brush seal, wherein

said inner circumferential brush seal comprises:

an attachment section provided at an outer circumferential side;

a brush portion in which a plurality of bristles are arranged from the attachment section to an inner circumferential direction; and

a free end portion which is an inner-circumferential-side end portion of the brush portion and is arranged opposite to an outer circumferential surface of the rotational shaft so as to contact or be adjacent to the outer circumferential surface, and

said outer circumferential brush seal comprises:

an attachment section provided at an outer circumferential side;

a brush portion in which a plurality of bristles are arranged from the attachment section to an inner circumferential direction; and

a free end portion which is an inner-circumferential-side end portion of the brush portion and is arranged opposite to an outer circumferential surface of the inner circumferential brush seal part so as to contact or be adjacent to the outer circumferential surface.

2. The brush seal device as set forth in claim 1, wherein at least one of said inner circumferential brush seal and said outer circumferential brush seal is a brush seal having a seal part and the free end portion formed on an inner circumferential side of a plurality of wire members bundled and arranged in a wall shape along a circumfer-

## 24

ence and formed into a ring shape by welding an outer end portion of the wire members combined to one another.

3. The brush seal device as set forth in claim 1, wherein at least one of said inner circumferential brush seal and said outer circumferential brush seal comprises a plurality of plate brush seal unit discs integrally formed by axially laminating themselves,

the plate brush seal unit disc comprises:

the attachment section;

the brush portion having narrow beams as the bristles extending from the attachment section to an inner circumferential side and inclined with respect to a circumferential direction; and

the free end portion.

4. The brush seal device as set forth in claim 1, wherein the brush portion of the inner circumferential brush seal and the brush portion of the outer circumferential brush seal are composed of bristles of the same material.

5. The brush seal device as set forth in claim 1, wherein the brush portion of the inner circumferential brush seal and the brush portion of the outer circumferential brush seal are composed of bristles of different materials.

6. The brush seal device as set forth in claim 1, wherein a plurality of the bristles of the inner-circumferential-side brush portion of the inner circumferential brush seal are arranged toward an inner circumferential direction to be inclined with respect to a radial direction,

a plurality of the bristles of the outer-circumferential-side brush portion of the outer circumferential brush seal are arranged toward an inner circumferential direction to be inclined with respect to a radial direction, and

the bristles of the inner circumferential brush seal and the bristles of the outer circumferential brush seal are inclined oppositely with respect to the radial direction.

7. The brush seal device as set forth in claim 1, further comprising:

a brush receiving part placed on the outer circumferential surface of the inner circumferential brush seal part so as to contact or be adjacent to the free end portion of the outer circumferential brush seal and to restrict a spreading out in an axial direction of the free end portion of the outer circumferential brush seal.

8. The brush seal device as set forth in claim 1, wherein the outer circumferential brush seal is formed to be inclined with respect to the axial direction of the rotational shaft.

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